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# WARTIME REPORT

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CHARTS FOR THE MINIMUM-WEIGHT DESIGN OF 24S-T ALUMINUM-ALLOY

FLAT COMPRESSION PANELS WITH LONGITUDINAL

Z-SECTION STIFFENERS

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# WASHINGTON

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# ADVANCE RESTRICTED REPORT

CHARTS FOR THE MINIMUM-WEIGHT DESIGN OF 2LS-T ALUMINUM-ALLOY
FLAT COMPRESSION PANELS WITH LONGITUDINAL

Z-SECTION STIFFENERS

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#### SULMARY

Design charts are developed for 245-T aluminum-allcy flat compression panels with longitudinal Z-section stiffeners. These charts make possible the design of the lightest panels of this type for a wide range of design requirements. Examples of the use of the charts are given and it is pointed out on the basis of these examples that, over a wide range of design conditions, the maintenance of buckle-free surfaces does not conflict with the achievement of high structural efficiency. The achievement of the maximum possible structural efficiency with 243-T aluminum-alloy panels, however, requires closer stiffener spacings than those now in common use.

#### INTRODUCTION

In a longitudinally stiffened compression panel, in which all the material is active in carrying load, the requirement of minimum weight is tantamount to that of carrying the load at the highest possible average stress. The average stress developed by such a panel under the loading conditions imposed is thus a direct measure of the structural efficiency of the panel. If longitudinally stiffened compression panels are to be designed for high structural efficiency without a large number of cut-and-try computations, it is desirable that design charts be prepared to indicate the average stress attainable under various loading conditions. The preparation of such charts requires that a suitable design parameter in which the important loading conditions are incorporated be found.

It has been found that a suitable parameter for longitudinally stiffened compression panels in the design of which the transverse stiffness can be neglected

is  $\frac{P_1}{L/\sqrt{c}}$ , where  $P_1$  is the compressive load per inch

of panel width, L is the panel length, or distance between supporting ribs, and c is the coefficient of end fixity at the ribs. The quantity P<sub>1</sub>, which is essentially independent of the distribution of material in the compression panel, can be estimated for a wing panel from the bending moment on the wing and the thickness and chord of the wing. The length L may be fixed by the presence of such installations as fuel tanks or armament or may be arbitrarily assigned for the purpose of arriving at a trial design.

In reference 2 buckling stresses were plotted against the parameter  $\frac{P_j}{L/\sqrt{c}}$ , with slightly different

notation, to form the basis of a theoretical study of the efficiencies of various types of stiffening elements. In the present paper the same parameter has been used as a basis for the preparation of design charts from extensive test data on 24S-T aluminum-alloy flat compression panels with longitudinal Z-section stiffeners; the data were obtained from reference 1 and from additional tests completed since publication of reference 1. These charts make possible the choice of the lightest panels of this type to conform to a wide range of design conditions. An appendix is presented in which the procedure followed in preparing the charts from test data is described and

the method for obtaining  $\frac{P_1}{L/\sqrt{c}}$  as a natural parameter

against which the average stress may be plotted to obtain a direct measure of structural efficiency is developed.

### SYMBOLS AND DEFINITIONS

The symbols used for the principal panel cross-sectional dimensions are indicated in figure 1. In addition, the following symbols are used:

- A<sub>1</sub> cross-sectional area per inch of panel width, or equivalent thickness of panel, inches
- L length of panel, inches
- P<sub>1</sub> compressive load per inch of panel width, kips per inch
- E modulus of elasticity in compression, ksi
- c coefficient of end fixity as used in Euler column formula
- k coefficient in formula for local-buckling stress
- ρ radius of gyration of panel cross section, inches
- reduction in effective modulus of elasticity when panel fails as a column beyond the elastic range
- σ<sub>cr</sub> critical stress, or stress for local buckling, ksi
- σc average stress at column failure, ksi
- omax average stress at local failure, ksi
- of average stress at failure for any panel, ksi

The average stress at which any particular panel fails,  $\overline{\sigma}_f$ , may be a local-failure stress, a column-failure stress, or the stress for a type of failure intermediate to these two. Failure by twisting of the stiffeners is included as a form of local failure. Because the design charts are based on actual test data, it is not necessary to make any distinction between local and twisting failure. Such a distinction, moreover, would be at best an arbitrary one, as the two types of failure are interrelated in the case of stiffened panels.

It should be noted that the local-failure stress  $\overline{\sigma}_{\text{max}}$ , which represents the maximum value of average stress that can be achieved in a given cross section as the panel length is reduced, is an average stress at <u>failure</u> and is not to be confused with the stress for local buckling  $\sigma_{cr}$ , which does not necessarily imply failure.

The term "local buckling" as used herein includes both buckling of the skin and buckling of the stiffeners, because neither of these elements can buckle without exerting moments on, and thus causing deformation of, the other element.

# DESIGN CHARTS

Design charts for 243-T aliminum-alloy flat compression panels with longitudinal Z-section stiffeners are presented in figures 2 to 5. The procedure used in the preparation of these charts from test data is described in the appendix. Values of A1/ts, necessary for arriving at a final design, are given in tables 1 to 3 for a wide range of dimension ratios.

In order to show the maximum stresses attainable by the use of panels of the type to which the charts apply, envelopes are indicated by the dashed lines for each value of the ratio  $b_S/t_S$  in figures 2 to 5. These envelopes have been combined (fig. 6) to give the over-all envelopes for the four values of the ratio  $t_W/t_S$ . The values of  $b_S/t_S$  and  $b_W/t_W$  needed in order that a panel will develop the stress indicated by an envelope are also given in figure 6.

The design parameter  $\frac{P_1}{L/\sqrt{c}}$ , against which stress is plotted in figures 2 to 6, comprises the principal design conditions: the compressive load per inch of panel width; the length of panel, or distance between supporting ribs; and the coefficient of end fixity. The most efficient (lightest) panel for a given combination of these conditions is that panel which will develop the highest average stress for the particular value of  $\frac{P_1}{L/\sqrt{c}}$ .

Discussion of charts. The charts include a wide range of panel proportions. All the charts have been drawn for a value of  $\frac{b_F}{b_W} = 0.1$ ; it is shown in the appendix (figs. 17 to 20), however, that curves for  $\frac{b_F}{b_W} = 0.3$  and 0.5 would be in close agreement

with the curves for  $\frac{b_F}{b_W} = 0.1$ . The curves of figures 2 to 5 may therefore be applied with reasonable accuracy for any value of  $b_F/b_W$  between 0.3 and 0.5. The available test data seem to indicate, moreover, that the most efficient use of material will be realized if a proportion in this range is selected. (See appendix.)

The short horizontal lines that intersect the curves of figures 2 to 5 indicate, for each panel cross section having appreciable local buckling, the stress at which this buckling occurs. In this report this stress is taken as that at which the compressive strain on one side of the skin or the stiffener web begins to be reduced with increasing load. This definition of buckling is convenient for structural testing; from the standpoint of aerodynamic smoothness, appreciable buckling probably takes place at stresses somewhat lower than those indicated on the charts. It will be noted that for some of the lower values of bs/ts and bw/tw no buckling stress is shown. In these cases, there will undoubtedly be some buckling but presumably it will occur at a stress coincident with or only very slightly below the failure stress.

It is pointed out that for  $\frac{t_W}{t_S} = 0.79$  and 1.00 (figs. 4 and 5), the curves for values of  $\frac{t_S}{t_S} = 25$  and 30

have been obtained entirely by extrapolation. These curves should therefore be used with a certain degree of caution. A few check tests made since the preparation of the charts, however, indicate that the curves will in no case be more than 6 percent unconservative. In all the other curves, it is believed that any unconservatism that may be present is of much smaller magnitude.

Discussion of tests and test panels. In order that the design charts may be properly used, it is necessary to know something of the test panels and the test results on which the design charts are based. The details of these tests are described in reference 1; some of the pertinent information regarding the tests follows:

The test panels consisted of six stiffeners and five bays. The panels were tested flat-ended and without edge support. A fixity coefficient of 3.75 was used in reducing the test data for application to an effective pin-ended length. The average compressive yield strength for the material of which the test panels were constructed was about 44 ksi; the minimum yield strength, about 41 ksi; and the maximum yield strength, about 46.5 ksi. The rivets were countersunk and were driven by the

NACA method of inserting a flat-head rivet from the stiffener side of the hole, upsetting the rivet shank into the countersumk cavity, and milling off the protruding portion of the upset shank. The rivets were ALTS-T (ANH42AD) and were of the sizes and spacings indicated by the following table:

tw ts	Rivet spacing ts	Rivet diameter ts
0.51	10.0	1.50
.63	12.3	1.94
.79	12.3	1.93
1.00	11.7	1.95

Because the compressive strength of stiffened panels may be affected by the size and spacing of the rivets used to attach stiffeners to skin (reference 3), the rivet attachment must be equivalent to that indicated by the foregoing table in order to be sure of realizing the strengths indicated by the design charts.

### USE OF DESIGN CHARTS AND EXAMPLES

If sheet material could be obtained in any desired thickness and if no special limitations were put on the design, it would be sufficient merely to find those proportions that would give the highest stress for the

given value of  $\frac{r_1}{L/\sqrt{c}}$ . Because certain limitations are usually imposed, however, the structure that represents the best compromise of all the requirements must be chosen.

The usual gages in which aluminum-alloy sheet is manufactured are such that if the four ratios of tw/ts in figures 2 to 6 are applied consecutively to a particular skin gage, the four stiffener gages that result will generally be consecutive standard gages. Interpolation between the curves of two consecutive charts (figs. 2 and 3, 3 and 4, etc.) is therefore unnecessary for most practical purposes.

The particular procedure to be used in obtaining a design from the charts will depend on the nature of the results desired. Three possible methods are discussed, and examples are given of designs obtained for a given load intensity and three different lengths by each of the methods.

The distinguishing features of each method are:

Ideal design

The method for obtaining the ideal design gives the lightest panel that could be obtained if the designer were not restricted to the use of standard sheet gages. The design is obtained by use of the over-all envelopes of figure 6 only.

Short method

The short design method provides, without lengthy computation, a near approach to the lightest panel that can be obtained by use of standard sheet gages. The design is obtained by use of the envelopes for given values of bs/ts that appear as dashed lines in figures 2 to 5.

Maximum efficiency

The method of designing for maximum structural efficiency gives the lightest panel that can be obtained by use of standard sheet gages. The design is obtained through a complete study of the individual solid curves in figures 2 to 5. The method is somewhat lengthy; examples have been worked out by its use, however, to serve as a check on the short method, so that that method can be used with confidence.

Each of the three methods is given as a series of steps for reaching the final designs. In the method for obtaining the ideal design, the detailed computations for the four values of  $t_W/t_S$  included in figure 6 are given for L=10, 20, and 30 inches with  $P_1=3.0$  kips per inch and c=1. In the other two methods, the detailed computations are given only for L=20 inches  $\frac{t_W}{t_S}=0.79$ , again with  $P_1=3.0$  kips per inch and c=1; final results are given, however, for the complete set of examples considered in the discussion of the first method. It is assumed in all cases that a skin thickness of 0.064 inch is necessary in order to comply with other design requirements. A value of  $t_S/t_S$ 0 of 0.4 is used throughout. In arriving at the final designs, no values of the dimension ratios outside of the ranges covered by the charts are given consideration.

Method for obtaining the ideal design. This method consists of picking from figure 6 the optimum proportions and the stress and computing from these the actual panel dimensions.

The values and computed quantities for the conditions previously mentioned are given in table 4 and are referenced to the steps in the following procedure:

(1) Compute 
$$\frac{P_1}{L/\sqrt{c}}$$
.

- (2) From the curves of figure 6 pick off for each value of  $t_W/t_S$  the values of  $b_S/t_S$ ,  $b_W/t_W$ , and  $\overline{\sigma}_f$  corresponding to the value of  $\frac{P_1}{L/\sqrt{c}}$ .
- (3) Pick from table 2 the values of  $A_1/t_S$  for the ratios determined in step 2. (If  $\frac{b_F}{b_W} = 0.3$  or 0.5 is used, table 1 or table 3, respectively, should be used instead of table 2.)
  - (lt) Compute

$$t_{S} = \frac{P_{1}}{\overline{\sigma}_{f} \frac{P_{1}}{\overline{t}_{S}}}$$

This formula is based on the equality

$$P_1 = \overline{\sigma}_f A_1$$

(5) Compute

$$t_{W} = \frac{t_{W}}{t_{S}} t_{S}$$

$$b_{S} = \frac{b_{S}}{t_{S}} t_{S}$$

$$b_{S} = \frac{b_{Y}}{t_{W}} t_{W}$$

This procedure results in four designs for each length, corresponding to the four values of tw/ts, for

the given conditions. (See table 4.) The values marked with footnote a in table 4 represent those chosen as approaching most closely the desired condition of ts = 0.064 inch; these values therefore give an indication of the proportions needed in a practical design to meet the design requirements most efficiently.

The resulting designs are shown as the ideal designs at the tops of figures 7 to 9, along with bar graphs of the average stress at failure and the buckling stress. The buckling stress for each design was obtained by interpolation from the short horizontal lines for buckling in figures 2 to 5. In some cases in which failure is by column action, the buckling stress shown by figures 2 to 5 will be greater than the failure stress for the designs obtained. Whenever this difference occurred in the present examples, the buckling stress is shown equal to the failure stress.

Short method for obtaining a practical design.— The short method consists of picking the optimum value of bw/tw and the corresponding stress for each value of bs/ts from the individual envelopes of figures 2 to 5 and computing from these values the actual panel dimensions. Panel designs that employ standard sheet gages are then selected from the various designs obtained.

The values and computed quantities for L = 20 inches and  $\frac{t_N}{t_S}$  = 0.79 are given in table 5 and are referenced to the steps in the following procedure:

- (1) Compute  $\frac{1}{L/\sqrt{c}}$ .
- (2) From the curves for a particular value of  $t_W/t_{S_2}$  (in this example, fig. 4 for  $t_S=0.79$  is used) pick off for each value of  $b_S/t_S$  the values of  $b_W/t_W$  (by interpolation along the dashed envelope) and  $\overline{o}_f$  (from the envelope) corresponding to the value of  $\frac{P_1}{L/\sqrt{c}}$ .
- (3) Pick from table 2 the values of  $A_1/t_S$  for the ratios determined in step 2.

# (4) Compute

$$t_{S} = \frac{P_{1}}{\overline{\sigma}_{f}} \frac{A_{1}}{t_{S}}$$

- (5) Plot b<sub>W</sub>/t<sub>W</sub>, t<sub>S</sub>, and  $\overline{\sigma}_f$  against b<sub>S</sub>/t<sub>S</sub> for the particular value of t<sub>W</sub>/t<sub>S</sub>. (The plot for the example being considered is shown in fig. 10.) Tabulate the values of b<sub>S</sub>/t<sub>S</sub>, b<sub>W</sub>/t<sub>W</sub>, and  $\overline{\sigma}_f$  corresponding to the point where t<sub>S</sub> equals the specified value.
- (6) Check computations by picking from table 2 the value of  $A_1/t_S$  corresponding to the ratios tabulated in step 5. If all computations and plots are correct,

$$P_1 = \overline{\sigma}_f \frac{A_1}{t_S} t_S$$

# (7) Compute

$$t_W = \frac{t_W}{t_S} t_S$$

$$b_{S} = \frac{b_{S}}{t_{S}} t_{S}$$

$$b_{W} = \frac{b_{W}}{t_{W}} t_{W}$$

(8) Repeat steps 2 to 7 for other values of tw/ts.

Like that for the ideal design, this procedure results, for each length considered, in one design for each value of  $t_{\rm W}/t_{\rm S}$ . It may not always be possible to find satisfactory designs under the conditions imposed for all values of  $t_{\rm W}/t_{\rm S}$ . (Note that no designs are given in figs. 8 and 9 for  $\frac{t_{\rm W}}{t_{\rm S}}=0.51$ .) All the designs resulting from the use of the short method utilize standard sheet gages and meet the requirement that  $t_{\rm S}=0.064$  inch. The choice of design now depends on arriving at a suitable

compromise between high stress and wide stiffener spacing.

If the prevention of buckling under load is considered important, then the buckling stress must also be taken into account in making a choice.

The designs obtained by carrying out the foregoing procedure for the several values of L and  $t_W/t_S$  are shown as the short-method designs in figures 7 to 9 along with bar graphs of the average stress at failure and the buckling stress.

Method of designing for maximum structural efficiency.—The maximum-efficiency method consists of computing the thickness required as bg/tg is varied for each value of bw/tw and selecting the designs for which the skin gage is equal to that desired. The procedure results in a series of possible designs for each value of tw/tg, from which those designs that provide the highest average stress at failure can be selected.

The values and computed quantities for L = 20 inches and  $\frac{t_{\rm N}}{t_{\rm S}}$  = 0.79 are given in table 6 and are referenced to the steps in the following procedure:

- (1) Compute  $\frac{P_1}{L/\sqrt{c}}$
- (2) From the curves for a particular value of  $t_W/t_S$  (in this example, fig. 4 for  $\frac{t_W}{t_S} = 0.79$  is used) pick off for each value of  $b_W/t_W$  and  $b_S/t_S$  the value of  $\overline{\sigma_f}$  corresponding to the value of  $\frac{P_1}{L/\sqrt{c}}$ .
- (3) Pick from table 2 the values of  $A_1/t_S$  corresponding to the ratios used in step 2.
  - (4) Compute

$$t_{S} = \frac{P_{1}}{\sigma_{f} \frac{A_{1}}{t_{S}}}$$

(5) Plot  $t_S$  and  $\overline{\sigma}_f$  against  $b_S/t_S$  for each value of  $b_W/t_W$  and  $t_W/t_S$ . Plot the particular value

of bw/tw at the value of bs/ts for which ts equals the specified value and mark the value of stress at that value of bs/ts. The plots of this step for the example under consideration are given in figure 11 as the short lines for the several values of bw/tw indicated. In order to avoid unnecessary confusion, only short portions of the curves, except the curve for  $\frac{b_w}{t_W} = 20$ , are shown.

- (6) After step 5 has been completed for all the values of  $b_W/t_W$ , draw curves of stress and of  $b_W/t_W$  against  $b_S/t_S$  through the points determined in step 5 (heavy curves in fig. 11).
- (7) Each of the curves drawn in step 6 represents a series of designs, all of which have the required value of  $t_S$  (in this case, 0.06; in.). The maximum point on the curve of  $\overline{\sigma}_f$  indicates the design for maximum structural efficiency for the particular value of  $t_W/t_S$ . Note this maximum value of  $\overline{\sigma}_f$ , the value of  $b_S/t_S$  at which it is reached, and the value of  $b_W/t_W$ , which can be picked from the curve of  $b_W/t_W$  against  $b_S/t_S$ .
- (8) Check computations by picking from table 2 the value of  $A_1/t_S$  corresponding to the ratios selected for maximum structural efficiency in step 7. If all computations and plots are correct,

$$P_1 = \overline{\sigma}_f \frac{A_1}{t_S} t_S$$

(9) Compute

$$t_{W} = \frac{t_{W}}{t_{S}} t_{S}$$

$$b_{S} = \frac{b_{S}}{t_{S}} t_{S}$$

$$b_{W} = \frac{b_{W}}{t_{W}} t_{W}$$

(10) Repeat steps 2 to 9 for other values of  $t_{\rm M}/t_{\rm S}$ .

This procedure results, for each length considered, in one design for each value of tw/ts. The choice of a design depends on arriving at a suitable compromise between high stress and wide stiffener spacing, with possible consideration for the buckling stress.

The designs obtained by carrying out the foregoing procedure for the several values of L and  $t_W/t_S$  are shown as the maximum-efficiency designs in figures 7 to 9 along with bar graphs of the average stress at failure and the buckling stress.

#### DISCUSSION

Figures 7 to 9 provide a visual comparison of the designs that result from use of the three methods presented. The short method of design gives in every case an average stress at failure very close to that obtained by designing on the basis of maximum structural efficiency; the buckling stress, however, is in some cases somewhat lower than that for the maximum-efficiency panel.

Whether the design obtained by the short method or the design for maximum efficiency is selected, the best design for  $P_1=3.0$  kips per inch, on the basis of stress, is obtained at L=10 inches with  $\frac{t_W}{t_S}=0.51$ , at L=20 inches with  $\frac{t_W}{t_S}=0.63$ , and at L=30 inches with  $\frac{t_W}{t_S}=0.79$ . In figure 6, however, the highest envelope, which gives the lightest design, is that for  $\frac{t_W}{t_S}=1.00$ . This apparent contradiction results from the fact that in working out the examples a skin thickness of 0.061 inch was specified. In order to reach the curve for  $\frac{t_W}{t_S}=1.00$  (fig. 6), a study of table 4 shows that the skin thickness would have to be 0.031 inch at L=10 inches, 0.011 inch at 20 inches, and 0.016 inch at 30 inches. Moreover, the stiffener spacings for designs having such small skin thicknesses are very small. (See table 4.) Because of limitations

on skin gages and stiffener spacings, therefore, it is frequently not possible to reach the envelope values of stress and hence the lowest possible weight.

Figures 7 to 9 show that the best panel (that with highest  $\overline{\sigma}_{r}$ ) obtained at each length by the maximumefficiency method does not buckle until failure or very close to failure. The best panel designed by the short method, although it may not have quite so high an average stress at failure as the maximum-efficiency panel, also does not buckle until very close to failure. This condition has been found to hold true over a wide range of design requirements. It is therefore evident that over a wide range of conditions the maintenance of bucklefree surfaces does not conflict with the achievement of high structural efficiency. The simultaneous achievement of both these ends by use of 24S-T aluminum-alloy panels, however, apparently requires closer stiffener spacings than those new in common use. For example, the maximum-efficiency designs for  $P_1 = 3.0$  kips per inch and to = 0.064 inch have the following spacings for the three lengths:

L (in.)	ts	b <sub>S</sub> (in.)
10	28.0	1.79
20	42.1	2.69
30	40.0	2.56

## CONCLUDING REMARKS

Charts are presented for the minimum-weight design of 245-T aluminum-alloy flat compression panels with longitudinal Z-section stiffeners. From examples based on the use of these charts, it is concluded that, over a wide range of design conditions, the maintenance of buckle-free surfaces on longitudinally stiffened compression panels does not conflict with the achievement of high structural efficiency. The achievement of the maximum possible structural efficiency with

24S-T aluminum-alloy panels, however, requires closer stiffener spacings than those now in common use.

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### APPENDIX

# METHOD OF PREPARATION OF DESIGN CHARTS

Development of design parameter  $\frac{P_1}{L/\sqrt{c}}$ . As stated

in the Introduction, the average stress developed by a longitudinally stiffened compression panel is a direct measure of the structural efficiency of the panel. It is further brought out that a suitable design parameter against which this average stress may be plotted

is  $\frac{1}{L/\sqrt{c}}$ , where  $P_1$  is the compressive load per inch of panel width, L is the panel length or distance between supporting ribs, and c is the coefficient of end fixity at the ribs.

The following derivation shows how the parameter  $\frac{1}{L/\sqrt{c}}$  evolves from the usual column formula:

The column formula may be written

$$\overline{\sigma}_{c} = \frac{\pi^{2} \tau E_{c}}{\left(\frac{L}{\rho \sqrt{c}}\right)^{2}} \tag{A1}$$

Multiplication and division of the right-hand side of equation (Al) by  $P_1^2$  gives

$$\overline{\sigma}_{c} = \pi^{2} \tau E_{c} \left(\frac{\rho}{F_{1}}\right)^{2} \left(\frac{P_{1}}{L/\sqrt{c}}\right)^{2}$$
(A2)

If the stiffened panel is to have a strength just equal to that required by the design conditions,  $P_1 = A_1 \sigma_c$  and equation (A2) may therefore be written

$$\overline{\sigma}_{c} = \pi^{2} \tau E_{c} \left(\frac{\rho}{A_{1}}\right)^{2} \left(\frac{P_{1}}{L/\sqrt{c}}\right)^{2} \left(\frac{1}{\overline{\sigma}_{c}}\right)^{2}$$

or

$$\frac{\overline{\sigma}_{c}^{3}}{\tau} = \pi^{2} E_{c} \left(\frac{\rho}{A_{1}}\right)^{2} \left(\frac{P_{1}}{L/\sqrt{c}}\right)^{2}$$

which may be written

$$\frac{\overline{\sigma}_{c}}{\sqrt[3]{T}} = \sqrt[3]{\pi^{2}E_{c}} \left(\frac{\rho}{A_{\perp}}\right)^{2/3} \left(\frac{P_{1}}{L/\sqrt{c}}\right)^{2/3} \tag{A3}$$

The quantity  $\sqrt[7]{\pi^2 E_C}$  in equation (A3) is fixed for a given material, as is the relationship between  $\overline{\sigma}_C$  and  $\tau$ , except for negligible shape effects. The quantity  $\frac{P_1}{L/\sqrt{c}}$  is the design parameter;  $\rho/A_1$  is dimensionless and is determined by the relative rather than the absolute dimensions of a panel. A plot of  $\overline{\sigma}_C$  against  $\frac{P_1}{L/\sqrt{c}}$  is therefore dependent on the ratios of the various panel dimensions and not on the absolute values of the dimensions.

Determination of average stress at local failure  $\overline{\sigma}_{max}$ . From equation (A3), the best panel of a given material for any value of  $\frac{P_1}{L/\sqrt{c}}$  on the basis of column strength apparently is that panel which has the highest value of  $\rho/A_1$ . Changes in proportions that result in an increase in  $\rho/A_1$  will, however, generally cause a decrease in the local-failure strength of the panel. (Local failure as used herein includes the phenomenon of twisting, which is in reality only a form of local failure that occurs when the lateral bending stiffness of the outstanding stiffener flange is relatively small.) The optimum panel for a particular application is given by the compromise of column and local-failure strengths that gives the highest stress at the given value of  $\frac{P_1}{L/\sqrt{c}}$ .

The value of the average stress at local failure  $\overline{\sigma}_{max}$ is difficult to determine theoretically. Certain test data are available, however, from reference 1 and from additional tests completed since the publication of reference 1. Those data that were obtained from the shortest panels of each cross section are summarized in figure 12, in which  $\sigma_{max}$  is plotted against  $t_W/b_W$  for various  $t_W/t_S$  and  $b_S/t_S$ . The ratio  $b_W/t_W$  has been values of inverted in this plot in order that the additional point  $\overline{\sigma}_{\text{max}} = 0$  when  $\frac{t_{yy}}{b_{\text{W}}} = 0$   $\left(\frac{b_{\text{W}}}{t_{\text{W}}} = \infty\right)$  might be used to aid in fairing curves through the test points. The plots of figure 12 make possible an interpolation of  $\overline{\sigma}_{max}$ between test points for intermediate values of the ratio by/tw. By plotting values of omax picked from the curves of figure 12 against  $t_S/o_S$ , values of  $\overline{\sigma}_{max}$ were also determined for intermediate values of bs/ts.

All the data shown in figure 12 are for a value of  $\frac{b_F}{b_W}=$  C.4. Test data for  $\frac{b_F}{b_W}=$  0.3 and 0.5, however, were also employed as a guide in fairing the curves, and the curves will be shown to be reasonably accurate for any value of  $b_F/b_W$  between 0.3 and 0.5.

Determination of stress for local buckling  $\sigma_{cr}$ . If the ranel did not buckle locally before failure, the theoretical results thus far precented, used in conjunction with values of  $\overline{\sigma}_{max}$ , would be sufficient to construct a design curve of  $\overline{\sigma}_{f}$  against  $\frac{P_{1}}{L/\sqrt{c}}$  for any panel. A typical curve for panels that do not buckle before failure is shown in figure 13. Unless the width-thickness ratios of the various plate elements of the panel are small or the panel is relatively long, however, there, will generally be some local buckling before failure. When this buckling takes place, the crosssectional moment of inertia of the ranel is reduced by the presence of ineffective areas; the original curve of column strength therefore no longer applies and the point at which brokling takes place must be connected with the line for local failure by means of a reduced curve. A typical curve, adjusted for the effects of local buckling is shown in figure 14.

The foregoing discussion shows that it is necessary to know the stress at which buckling takes place. Data on buckling stresses from reference 1 plus additional data now available are therefore plotted in figure 15

for  $\frac{b_F}{b_W} = 0.4$ . Because the measured value of b/t for the element (skin or stiffener web) that first showed buckling in a test panel was never in exact agreement with the specified nominal value, the observed buckling stresses from reference 1 were corrected for use in figure 15 according to the following formula:

$$(\sigma_{cr})_{corrected} = (\sigma_{cr})_{observed} \frac{\left(\frac{b}{t}\right)^{2}_{measured}}{\left(\frac{b}{t}\right)^{2}_{nominal}}$$

where the value of b/t is that for the web of the stiffener or for the skin between stiffeners, depending on which of these elements first gave evidence of buckling. This correction formula is based on the fact that, other factors being equal, the critical stress is inversely proportional to the square of the width-thickness ratio. No account is taken herein of the fact that this relationship is not entirely true for stresses beyond the elastic range; it is assumed that neglecting this fact will have no significant effect because the total correction is relatively small.

The method used in fairing curves through the test points in figure 15 is as follows:

For the horizontal portions of the curves on the right-hand side of figure 15, the skin is primarily responsible for the buckling; the ordinates for the curves in this region are determined by drawing average lines through the test points. As the value of  $t_W/b_W$  is reduced, however, the responsibility for the buckling shifts to the stiffeners and there is a reduction in  $c_{\rm cr}$ . In the absence of adequate test data for low values of  $t_W/b_W$ , certain theoretical considerations are used for determining the values of  $c_{\rm cr}$  in this region.

It is possible to describe certain limiting conditions that determine curves between which the correct curves must lie. As the value of  $t_W/b_W$  approaches zero, with all other dimension ratios held constant, the skin tends to become infinitely stiff by comparison with the stiffener and the stiffener approaches a condition of complete fixity at the edge where it is attached to the skin. This condition of complete fixity represents the upper limit of buckling stress. The value of k, the coefficient in the formula for local-buckling stress (reference 4), when applied to the stiffener web may be taken for this condition as the geometric mean of the value of k for the web of a Z-section column  $\frac{b_F}{b_{cc}} = 0.4$  (about 3.77, see reference 4) and the

value of k for a flat plate fixed at both edges (about 6.98, see reference 5). This value of k is  $\sqrt{3.77} \times 6.98$ , or 5.13. The upper dashed curve in figure 15 gives  $\sigma_{\rm cr}$  for k = 5.13. The use of the geometric mean of values of k to obtain the critical stress for a plate with different restraints along the two unloaded edges is discussed and justified for practical use in reference 5.

When  $\frac{b_{ij}}{t_{ij}} = \frac{b_{ij}}{t_{ij}}$ , it is a reasonable and probably

conservative assumption to consider the stiffener hinged at the edge where it is attached to the skin. This hinged condition represents the lower limit of buckling stress. The value of k for the web of the stiffener may be taken for this condition as the geometric mean of 3.77 for the simple Z-section and the value for a flat plate hinged at both edges (4.00, see reference 5) or  $k = \sqrt{3.77} \times 4.00 = 3.88$ . The lower dashed curve in figure 15 gives  $\sigma_{\rm cr}$  for k = 3.98. In the preparation of the two dashed curves, the effect of reduction in the modulus of elasticity for stresses beyond the elastic range was determined from figure 8 of reference 6.

The solid curve on the left-hand side of figure 15 is drawn in to give a gradual transition from the lower dashed curve in the region where  $\frac{b_W}{t_W} = \frac{b_S}{t_S} \quad \text{toward the unper dashed curve as } t_W/b_W \quad \text{approaches zero.} \quad \text{In the}$ 

region where  $\frac{b_W}{t_W} = \frac{b_S}{t_S}$  the curves are faired into the horizontal lines drawn through the test points. A single curve was considered sufficient for all values of  $t_W/t_S$  for the left-hand portion of figure 15, because the few test points that were available in this region indicated that the individual curves would be so close together as to be almost indistinguishable.

The curves of figure 15, like those of figure 12, were cross-plotted to give buckling stresses for the intermediate values of bg/tg that appear in figures 2 to 5.

Preparation of final curves. The procedure used in the preparation of the final curves of figures 2 to 5 is illustrated in figure 16. An outline of this procedure is as follows:

- (1) Draw curve for column strength corresponding to the value of  $\rho/A_1$  for the panel cross section. For the curves of this report, the column curve for  $2\mu$ S-T aluminum alloy was obtained from equations (5) and (6) and table I, all of reference 7.
- (2) Plot the values of stress for local buckling and for local failure of panel obtained from the cross plots of the curves in figures 12 and 15.
- (3) Plot available test data and fair curves between buckling stress and local-failure stress. This fairing was done first for those curves for which test data were available; the remaining curves were then faired in a manner consistent with the curves already established.

In a few cases (low  $b_S/t_S$  with high  $b_W/t_W$ ) the test data indicated that the curves did not follow the smooth transition between column and local failure indicated by figure 16. Instead the curves tended to bend over sharply, in some cases even below the buckling stress given by figure 15, and to follow very nearly a straight line up to the average stress for local failure. No explanation is offered for this phenomenon; the available test data were used as the sole guide for fairing the curves in these cases.

Correlation between design curves and test data .-The test data of reference 1 as well as the additional data made available since the publication of reference 1 are plotted against the parameter in figures 17 to 20. Appropriate curves taken from figures 2 to 5 are also drawn in these figures and good agreement between the final design curves and the test data for exists throughout the range of the data. In order to make it possible, if desired, to check the correlation on  $\frac{b_{\rm F}}{b_{\rm W}} = 0.3, 0.4,$ a larger-scale plot, the test data for and 0.5 are given in table 7 in a form suitable for plotting directly on the design charts (figs. 2 to 5). Table 7 and figures 17 to 20 also make it possible to determine in which regions the design charts are substantiated by test data and in which regions they were obtained by interpolation or extrapolation.

Figures 17 to 20 indicate that there would be little difference in the curves for  $\frac{b_F}{b_W}=0.3$ , 0.4, and 0.5 but that the curves for  $\frac{b_F}{b_W}=0.2$  and probably 0.7 would be lower than those for  $\frac{b_F}{b_W}=0.4$ . The most efficient use of material will therefore be realized if a value of  $b_F/b_W$  between 0.3 and 0.5 is used. It is for this range that the design charts are intended to be used, although they are based on the specific data for  $\frac{b_F}{b_W}=0.4$ .

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TABLE 1

VALUES OF 
$$A_1/t_S$$
 FOR FLAT PANELS WITH Z-SECTION STIFFENERS.  $\frac{b_P}{b_W} = 0.3$ .
$$\begin{bmatrix} \frac{A_1}{t_S} = 1 + \frac{b_W}{t_W} \left(1 + \frac{b_P}{b_W}\right) + \frac{b_A}{t_W} - \left(2 - \frac{\pi}{2}\right) \left(\frac{r_A}{t_W} + \frac{r_P}{t_W} + 1\right) \left(\frac{t_W}{t_S}\right)^2 \end{bmatrix}$$

 $\frac{t_{W}}{t_{Q}} = 0.51$ 

										<b>5</b>											
b <sub>W</sub> /t <sub>W</sub>	20	21	22	23	ᆲ	25	26	27	28	29	30	32	34	36	38	40	142	孙	146	48	50
J 50	1.340 1.327 1.316 1.305 1.294	1.353 1.340 1.328 1.316 1.306	1.366 1.352 1.340 1.328 1.317	1.379 1.365 1.352 1.340 1.328	1.392 1.377 1.364 1.351 1.340	1.405 1.390 1.376 1.363 1.351	1.418 1.402 1.388 1.375 1.362	1.431 1.415 1.400 1.386 1.373	1.444 1.427 1.412 1.398 1.385	1.457 1.440 1.424 1.410 1.396	1.470 1.452 1.436 1.421 1.407	1.496 1.477 1.460 1.445 1.430	1.522 1.502 1.485 1.468 1.452	1.548 1.528 1.509 1.491 1.475	1.574 1.553 1.533 1.514 1.497	1.600 1.578 1.557 1.538 1.520	1.651 1.626 1.603 1.581 1.561 1.542	1.652 1.628 1.605 1.584 1.565	1.678 1.653 1.629 1.608 1.587	1.704 1.678 1.654 1.631 1.610	1.730 1.703 1.678 1.654 1.633
31 32 33 34 35	1.285 1.276 1.268 1.260 1.252	1.296 1.287 1.278 1.270 1.262	1.307 1.297 1.288 1.280 1.272	1.318 1.308 1.298 1.290 1.281	1.329 1.318 1.309 1.300 1.291	1.340 1.329 1.319 1.310 1.301	1.350 1.359 1.329 1.320 1.310	1.361 1.350 1.339 1.329 1.320	1.372 1.361 1.350 1.339 1.330	1.383 1.371 1.360 1.349 1.339	1.394 1.382 1.370 1.359 1.349	1.416 1.403 1.391 1.379 1.368	1.438 1.424 1.411 1.399 1.388	1.459 1.445 1.432 1.419 1.407	1.481 1.466 1.452 1.439 1.426	1.503 1.487 1.473 1.459 1.446	1.525 1.509 1.493 1.479 1.465	1.547 1.530 1.514 1.498 1.484	1.569 1.551 1.534 1.518 1.504	1.590 1.572 1.555 1.538 1.523	1.612 1.593 1.575 1.558 1.542
39	11.227	11.235	بستحادا	1.255	11.201	1.270	11.279	11.287	11.296	11.505	1.515	11.551	11.348	11.565	11.385	11.400	1.452 1.440 1.428 1.417 1.407	11.435	1.452	11.469	11.M37 I
146	1.192 1.184	1.199	1.207	1.214	1.221	1.229	1.236	1.214 1.233	1.251	1.258	1.256	1.269	1.295	1.310	1.324	1.339	1.387 1.370 1.354 1.339 1.325	1.353	1.367	1.398	1.413
54 56 58	1.164 1.158 1.152	1.170 1.164 1.158	1.176 1.170 1.164	1.182 1.176 1.170	1.189 1.182 1.176	1.195 1.198 1.181	1.201 1.194 1.187	1.207 1.200 1.193	1.214	1.220 1.212 1.205	1.226 1.218 1.211	1.239 1.230 1.222	1.251 1.242 1.234	1.264 1.254 1.246	1.276 1.266 1.257	1.289 1.279 1.269	1.313 1.301 1.291 1.281 1.271	1.314	1.326 1.315 1.304	1.339 1.327 1.316	1.351 1.339 1.327
65 70 75	1.136 1.126 1.118	1.141 1.131 1.122	1.146 1.136 1.127	1.152 1.141 1.131	1.157 1.146 1.136	1.162 1.150 1.140	1.167 1.155 1.145	1.172 1.160 1.149	1.178 1.165 1.154	1.183 1.170 1.158	1.188 1.175 1.163	1.198 1.184 1.172	1.209 1.194 1.181	1.219 1.203 1.190	1.230 1.213 1.199	1.240 1.223 1.208	1.250 1.232 1.217	1.261 1.242 1.226	1.271 1.252 1.235	1.282 1.261 1.244	1.292 1.271 1.253

TABLE 1  $\frac{b_F}{b_W} = 0.3 - \text{Continued}$ 

 $\frac{t_{W}}{t_{S}} = 0.63$ 

b <sub>W</sub> /t <sub>W</sub>	20	21	22	23	214	25	26	27	28	29	30	32	34	36	38	40	42	1414	46	48	50
25 26 27 28 29 30	1.531 1.511 1.492 1.474 1.458 1.443	1.552 1.531 1.511 1.493 1.476 1.460	1.573 1.551 1.530 1.511 1.494 1.477	1.593 1.570 1.549 1.530 1.511 1.494	1.614 1.590 1.568 1.548 1.529 1.512	1.635 1.610 1.588 1.567 1.547 1.529	1.655 1.630 1.607 1.585 1.565	1.676 1.650 1.626 1.603 1.583 1.563	1.696 1.670 1.645 1.622 1.600 1.580	1.717 1.690 1.664 1.640 1.618 1.598	1.738 1.709 1.683 1.659 1.636 1.615	1.779 1.749 1.721 1.696 1.672 1.649	1.820 1.789 1.760 1.732 1.707 1.684	1.862 1.828 1.798 1.769 1.743 1.718	1.903 1.868 1.836 1.806 1.778 1.752	1.944 1.908 1.874 1.843 1.814 1.787	1.985 1.948 1.912 1.880 1.849 1.821	2.027 1.987 1.951 1.917 1.885 1.856	2.068 2.027 1.989 1.954 1.921 1.890	2.109 2.067 2.027 1.990 1.956 1.924	2.151 2.106 2.065 2.027 1.992 1.959
<u>왕</u> 55	1.391	1.406	1.421	1.436	1.451	1.467	1.468	1.497	1.497	1.512	1.527	1.573	1.586	1.615	1.645	1.674	1.704	1.755	1.763	1.792	1.822
																					1.799 1.777 1.757 1.738 1.719
50	1.266	1.276	1.286	1.297	1.307	11.217	1.520	1.000	1,540	1.000	1.509	1.590	1.410	1.451	1.451	1.472	1.495	1.513	1.004	4.777	
52 556 58 60	1.255 1.246 1.237 1.229 1.221	1.265 1.256 1.246 1.238 1.230	1.275 1.265 1.256 1.247 1.239	1.285 1.275 1.265 1.256 1.247	1.295 1.284 1.274 1.265 1.256	1.305 1.294 1.283 1.273 1.264	1.315 1.303 1.292 1.282 1.273	1.325 1.313 1.302 1.291 1.282	1.335 1.322 1.311 1.300 1.290	1.345 1.332 1.320 1.309 1.299	1.355 1.342 1.329 1.318 1.307	1.375 1.361 1.348 1.336 1.325	1.394 1.380 1.366 1.354 1.342	1.414 1.399 1.385 1.371 1.359	1.434 1.418 1.403 1.389 1.376	1.454 1.437 1.421 1.407 1.393	1.474 1.456 1.440 1.425 1.411	1.494 1.475 1.458 1.443 1.428	1.494 1.477 1.460 1.445	1.533 1.514 1.495 1.478 1.462	1.553 1.533 1.534 1.496 1.479
65 70 75	11.190	11.197	1.220 1.205 1.191	11.212	11.219	11.227	11.234	11.241	11.2h9	J1.256	11.263	11.278	11.293	11.308	J1.322	11.337	11.352	11.367	11.381	11.396	11.411

TABLE 1

$$\frac{b_{\mathbf{p}}}{b_{\mathbf{W}}} = 0.3$$
 - Continued

$$\frac{t_{\overline{W}}}{t_{\overline{S}}} = 0.79$$

bg/tg	20	21	22	23	21,	25	26	27	28	29	<b>3</b> 0	32	刄	36	<b>3</b> 8	40	42	Щ	46	<b>48</b>	50
26 27 28 29	1.777 1.748 1.721 1.697	1.808 1.778 1.750	1.839 1.808 1.779	1.871 1.838 1.808	1.938 1.902 1.868 1.837 1.808 1.781	1.933 1.898 1.866	1.964 1.928 1.895	1.995 1.958 1.924 1.892	2.027 1.989 1.953 1.920	2.058 2.019 1.982 1.948	2.049 2.049 2.011 1.976	2.151 2.109 2.069 2.032	2.214 2.169 2.127 2.088	2.276 2.229 2.185 2.144	2.339 2.289 2.243 2.200	2.401 2.301 2.301 2.256	2.463 2.409 2.359 2.312	2.526 2.469 2.417 2.368	2.588 2.529 2.475 2.121	2.651 2.590 2.533 2.480	2.713 2.650 2.590 2.536
34 35	1.577	1.600	1.623	1.647	1.756 1.733 1.710 1.690 1.670	1.693	1.716	1.739	1.763	1.786	1.809	1.855	1.902	1.948	1.994	2.041	2.087	2.157	2.215	2.226	2.273
1 28	13.5321	1.553	11.576	11.596	1.651 1.634 1.617 1.601 1.586	11.6381	1,6601	1.681	1.702	1:72h	1,71,5	11.788	11.830	1.873	1.916	1.959	12.001	120146	2.087	12.129	12.1721
146	1.439	1.457	1.474	1.492	1.558 1.533 1.510 1.488 1.469	1.527	1.545	1.563	1.556	1.598	1.615	1.651	1.686	1.721	1.757	1.792	1.827. 1.793	1.862	1.898	1.894	1.928
54 56 58	1.374 1.361	1.389	1.404 1.390	1.419 1.404	1.451 1.434 1.419 1.404 1.391	1.449 1.433	1.464 1.448 1.432	1.479 1.462 1.466	1.494 1.477 1.460	1.509 1.491 1.47h	1.524 1.506 1.488	1.554 1.535 1.516	1.584 1.564 1.564	1.614 1.593 1.572	1.645 1.621 1.600	1.675 1.650 1.628	1.705 1.679	1.735 1.708 1.684	1.765 1.737 1.712	1.795 1.766 1.760	1.825 1.795 1.768
1 70 i	1.289	1.300	1.312	1.323	1.361 1.335 1.313	1.347	1.358	1.370	1.381	1.393	1.404	1.4281	1.451	1.474	1.497	1.520	11.544.	1.567	1.590	1.613	1.6361

TABLE 1
$$\frac{b_F}{b_W} = 0.3 - Concluded$$

$$\frac{t_W}{t_S} = 1.00$$

bw/tw	20	21	22	23	ᆀ	25	26	27	28	29	50	32	잿	36	<b>3</b> 8	140	42	排	46	<b>ц</b> 8	50
25 26 27 28 29 30	2.247 2.199 2.154 2.113 2.075 2.039	2.299 2.249 2.202 2.160 2.120 2.082	2.351 2.299 2.251 2.206 2.164 2.126	2.403 2.349 2.299 2.252 2.209 2.169	2.455 2.399 2.347 2.254 2.254 2.212	2.506 2.449 2.395 2.345 2.256	2.559 2.499 2.443 2.592 2.344 2.299	2.611 2.549 2.491 2.438 2.389 2.342	2.663 2.599 2.540 2.485 2.433 2.386	2.715 2.649 2.588 2.531 2.478 2.429	2.767 2.699 2.636 2.577 2.523 2.472	2.871 2.799 2.732 2.670 2.613 2.559	2.975 2.899 2.828 2.763 2.702 2.646	3.079 2.999 2.925 2.856 2.792 2.732	5.185 5.099 5.021 2.949 2.882 2.819	3.287 3.199 3.117 3.042 2.971 2.906	3.291 3.299 3.214 3.135 3.061 2.992	3.495 3.399 3.310 3.227 3.151 3.079	3.599 3.499 3.406 3.320 3.240 3.166	3.703 3.599 3.502 3.413 3.330 3.252	3.807 3.699 3.599 3.506 3.420 3.339
31 32 33 34 35	2.005 1.974 1.944 1.917 1.890	2.047 2.015 1.984 1.955 1.928	2.089 2.055 2.023 1.993 1.965	2.131 2.096 2.063 2.031 2.002	2.173 2.136 2.102 2.070 2.039	2.215 2.177 2.141 2.078 2.076	2.257 2.218 2.181 2.146 2.113	2.299 2.258 2.220 2.184 2.150	2.341 2.299 2.260 2.223 2.188	2.383 2.340 2.299 2.261 2.225	2.425 2.380 2.338 2.299 2.262	2.509 2.461 2.417 2.376 2.336	2.592 2.543 2.496 2.452 2.410	2.676 2.624 2.575 2.528 2.485	2.760 2.705 2.654 2.605 2.559	2.844 2.786 2.732 2.681 2.633	2.928 2.868 2.811 2.758 2.708	3.012 2.949 2.890 2.834 2.782	3.096 3.030 2.969 2.911 2.856	3.180 3.111 3.048 2.987 2.930	3.263 3.193 3.126 3.064 3.005
39	1.799	11.832	1.866	1.899	1.932	1.966	1.999	12.033	2.066	12.099	12.133	12.199	2,266	2.333	2.399	2.466	12.533	12.599	2.666	12.733	2.949 2.896 2.847 2.799 2.754
46 48 50	1.678 1.649 1.623	1.706 1.676 1.649	1.734 1.703 1.675	1.762 1.731 1.701	1.791 1.758 1.727	1.819 1.785 1.753	1.812 1.779	1.875 1.839 1.805	1.904 1.866 1.831	1.932 1.893 1.857	1.960 1.920 1.883	2.017 1.974 1.935	2.073 2.028 1.987	2.130 2.083 2.039	2.186 2.137 2.091	2.243 2.191 2.143	2.299 2.245 2.195	2.356 2.299 2.247	2.412	2.469 2.408 2.351	2.671 2.595 2.525 2.462 2.403
52 51 56 58 60	1.599 1.577 1.557 1.537 1.519	1.624 1.601 1.580 1.560 1.541	1.649 1.625 1.603 1.582 1.563	1.674 1.649 1.626 1.605 1.584	1.699 1.673 1.649 1.627 1.606	1.724 1.698 1.673 1.649 1.628	1.749 1.722 1.696 1.672 1.649	1.774 1.746 1.719 1.694 1.671	1.799 1.770 1.742 1.717 1.693	1.824 1.794 1.765 1.739 1.714	1.849 1.818 1.789 1.761 1.736	1.899 1.866 1.835 1.806 1.779	1.949 1.914 1.882 1.851 1.823	1.999 1.962 1.928 1.896 1.866	2.049 2.011 1.974 1.941 1.909	2.099 2.059 2.021 1.986 1.953	2.149 2.107 2.067 2.030 1.996	2.199 2.155 2.114 2.075 2.040	2.249 2.203 2.160 2.120 2.085	2.299 2.251 2.207 2.165 2.126	2.349 2.299 2.253 2.210 2.170
65 70 75	1.479 1.445 1.416	1.499 1.464 1.433	1.519 1.482 1.450	1.539 1.501 1.468	1.559 1.520 1.485	1.579 1.538 1.502	1.599 1.557 1.520	1.619 1.575 1.537	1.639 1.594 1.554	1.659 1.612 1.572	1.679 1.631 1.589	1.719 1.668 1.624	1.760 1.705 1.658	1.800 1.742 1.693	1.840 1.780 1.728	1.880 1.817 1.762	1.920 1.854 1.797	1.960 1.891 1.832	2.000 1.928 1.866	2.040 1.965 1.901	2.080 2.002 1.936

TABLE 2

VALUES OF A<sub>1</sub>/t<sub>3</sub> FOR FLAT PANELS WITH Z-SECTION STIFFENERS. 
$$\frac{b_F}{b_W} = 0.4.$$

$$\begin{bmatrix} \frac{A_1}{t_S} = 1 + \frac{b_W}{t_W} \left(1 + \frac{b_F}{b_W}\right) + \frac{b_A}{t_W} - \left(2 - \frac{w}{2}\right) \left(\frac{r_A}{t_W} + \frac{r_F}{t_W} + 1\right)}{b_S/t_S} \left(\frac{t_W}{t_S}\right)^2 \end{bmatrix}$$

$$\frac{t_W}{t_S} = 0.51$$

by/ty	20	21	22	23	24	25	26	27	28	29	30	32	34	36	38	40	þг	枡	46	48	50
25 26 27 28 29 30	1.525	1.547	1.348	1.575	1.432 1.416 1.400 1.386 1.373 1.360	1.385	1.398	1.425	1.423	1.451	1.448	1.490	1.498	1.542	1.549	1.5%	1.599	1.624	1.649	1.674	11.7241
31 32 33 34 35	1.292 1.283 1.275 1.267	1.294 1.294 1.286 1.278	1.315 1.306 1.297 1.288	1.326 1.317 1.307 1.298	1.509	1.349 1.339 1.329 1.319	1.350 1.359 1.339	1.372 1.361 1.350 1.340	1.363 1.372 1.361 1.350	1.395 1.383 1.372 1.361	1.406 1.394 1.382 1.371	1.429 1.416 1.404 1.392	1.452 1.438 1.425 1.413	1.474 1.460 1.446 1.434	1.497 1.482 1.468 1.455	1.520 1.504 1.489 1.475	1.543 1.526 1.511 1.496	1.548 1.532 1.537	1.588 1.570 1.554 1.538	1.592	1.614 1.596 1.579
36 37 38 39 40	1.246	1.256	1.272 1.265 1.259	1.282 1.275 1.268	1.300 1.292 1.285 1.277 1.270	1.294	1.304 1.296	1.313	1.323	1.332 1.324	1.342	1.361 1.352	1.380	1.399	1.430 1.419 1.408	1.450 1.458 1.427	1.457 1.457 1.445	1.476	1.495 1.483	1.514	احجروها
12 146 150	1.203	1.211	1.219	1.227	1.257 1.246 1.235 1.225 1.216	1.243	1.251	1.248	1.257	1.275	1.283	1.298	1.301	1.330	1.346	1.362	1.362	1.393	1.409	1.444 1.425 1.407	1.483 1.461 1.441 1.422 1.406
3.45880 3.46880	1.161	1.168	11.174	1.180	1.186	11.193	1.199	11.205	1.212	1.218	1.224	1.237	1.249	1.262	1.274	1.287	1.299	1.512	1.524	11.557	1.390 1.376 1.362 1.350 1.338
65 70 75	1.136	1.139	1.366	1.149	1.166 1.154 1.144	11.160	1.165	1.170	1.175	l1.180	11.186	1.196	1.206	1.217	1.227	1.238	1.248	1.258	1.269	11.279	1.512 1.290 1.270

TABLE 2 
$$\frac{b_{\mathbf{F}}}{b_{\mathbf{W}}} = 0.14 - Continued$$

$$\frac{t_{W}}{t_{S}} = 0.63$$

bg/tg	20	21	22	23	24	25	26	27	28	29	30	32	刄	36	38	40	42	种	46	148	50
25 26 27 28	1.485	1.505	1.52	1.543	1.562	1.581	1.622	1.620	1.652	1.658	1.701	1.741	1.754	1.820	1.850	1.869	1.939	1.945	1.984	2.059	2.230 2.183 2.139 2.098 2.060 2.025
31 32 33 34 35	13 .3.33	しょしょろの	しょっしんかい	1163	11 -1:79	1.496	11.512	11.528	11.545	11.561	11.577	11.610	11.643	11.676	11.708	11.741	11.771	11.806	11.839	11.872	1.992 1.961 1.932 1.904 1.879
38 39 40	1.380 1.370 1.361 1.352	1.395 1.385 1.375 1.366	1.411 1.400 1.389 1.380	1.426 1.444 1.404 1.394	1.441 1.429 1.418 1.408	1.456 1.444 1.432 1.421	1.471 1.458 1.446 1.435	1.486 1.473 1.461 1.449	1.487 1.475 1.463	1.516 1.502 1.489 1.477	1.531 1.517 1.503 1.491	1.546 1.532 1.519	1.591 1.575 1.560 1.546	1.574	1.651 1.634 1.617 1.602	1.681 1.663 1.646 1.630	1.711 1.692 1.674 1.658	1.741 1.721 1.703 1.685	1.771 1.751 1.731 1.713	1.780 1.760 1.741	1.809 1.788 1.769
12 11 16 18 50	1.335 1.320 1.306 1.293 1.282	1.348 1.333 1.318 1.305 1.293	1.362 1.345 1.330 1.316 1.304	1.375 1.358 1.342 1.328 1.315	1.388 1.370 1.354 1.340 1.326	1.401 1.383 1.366 1.351 1.337	1.415 1.396 1.379 1.363 1.348	1.428 1.408 1.391 1.374 1.359	1.441 1.421 1.403 1.386 1.370	1.454 1.434 1.415 1.397 1.382	1.467 1.446 1.427 1.409 1.393	1.494 1.471 1.451 1.432 1.415	1.520 1.497 1.475 1.455 1.437	1.547 1.522 1.499 1.479 1.459	1.573 1.547 1.523 1.502 1.482	1.600 1.573 1.548 1.525 1.504	1.626 1.598 1.572 1.548 1.526	1.653 1.623 1.596 1.571 1.548	1.679 1.648 1.620 1.594 1.571	1.706 1.674 1.614 1.617 1.593	1.732 1.699 1.668 1.641 1.615
52 51 56 58 60	1.261	1.271	1.281	1.292	1.302	1.312	1.322	1.333	1.343	1.353	1.364	1.384	1.405	1.425	1.446	1.467	1.487	1.508	1.528	1.549	1.591 1.569 1.549 1.530 1.512
65 70 75	11.201	11.209	11,217	11.225	11.233	11.241	11.249	11.257	11.265	11.273	11.280	11.296	11.312	11.328	11.344	11.360	11.376	11.392	11.408	11.423	1.473 1.439 1.410

TABLE 2  $\frac{b_F}{b_W} = 0.4 - Continued$ 

$$\frac{t_W}{t_S} = 0.79$$

b <sub>3</sub> /t <sub>3</sub>	20	21	22	23	24	25	26	27	28	29	30	32	34	36	38	40	142	ݪݪ	46	148	50
1 29	11.740	11.770	11.800	11.830	11.860	11.890	11.920	11.950	11.981	12.011	12.0 <u>u</u> 1	12.101	2.347 2.295 2.247 2.203 2.161 2.123	12.222	12.262	12.502	12.402	12.465	12.523	12.585	2.906 2.833 2.765 2.702 2.643 2.589
35	1.631	1.657 1.638	1.663	1.688	1.713	1.759	1.763	1.788	1.836	1.862	1.862	1.939	1.962	2.012	2.062	2.112	2.196	2.218	2.262	2.350	2.362
37 38 39 40	1.580 1.564 1.550 1.536	1.603 1.587 1.572 1.558	1.627 1.610 1.595 1.580	1.650 1.633 1.617 1.602	1.674 1.656 1.640 1.624	1.698 1.679 1.662 1.645	1.721 1.702 1.684 1.667	1.745 1.725 1.707 1.689	1.769 1.748 1.729 1.711	1.792 1.771 1.752 1.733	1.816 1.794 1.774 1.755	1.863 1.840 1.819 1.798		1.957 1.932 1.908 1.886	2.005 1.978 1.953 1.929	2.052 2.024 1.998 1.973	2.099 2.070 2.043 2.017	2.146 2.116 2.088 2.060	2.194 2.162 2.132 2.104	2.241 2.208 2.177 2.148	2.288 2.254 2.222 2.192
14 14 14 19 50	1.511 1.487 1.466 1.447 1.429	1.531 1.507 1.485 1.465 1.446	1.552 1.527 1.504 1.483 1.464	1.573 1.547 1.523 1.501 1.481	1.594 1.567 1.542 1.520 1.499	1.615 1.587 1.561 1.538 1.516	1.635 1.607 1.580 1.556 1.534	1.656 1.626 1.599 1.574 1.551	1.677 1.646 1.618 1.592 1.569	1.698 1.666 1.637 1.611 1.586	1.719 1.686 1.656 1.629 1.604	1.760 1.726 1.694 1.665 1.639	1.802 1.765 1.732 1.702 1.674	1.844 1.805 1.770 1.738 1.709	1.885 1.845 1.808 1.774 1.743	1.927 1.885 1.846 1.811 1.778	1.968 1.924 1.884 1.847 1.813	2.010 1.964 1.922 1.884 1.848	2.052 2.004 1.960 1.920 1.883	2.093 2.043 1.998 1.956 1.918	2.135 2.083 2.036 1.993 1.953
54 56 58	1.397 1.383 1.370	1.413 1.399 1.385	1.430 1.414 1.400	1.446 1.430 1.415	1.462 1.445 1.430	1.478 1.461 1.445	1.494 1.477 1.460	1.492 1.475	1.527 1.508 1.490	1.543 1.523 1.505	1.559 1.539 1.520	1.591 1.570 1.551	1.624 1.601 1.581	1.656 1.633 1.611	1.688 1.664 1.641	1.721 1.695 1.671	1.753 1.726 1.701	1.786 1.757 1.731	1.818 1.789 1.761	1.850 1.820 1.792	1.917 1.883 1.851 1.822 1.794
65 70 75	1.330 1.306 1.286	1.343 1.319 1.298	1.357 1.331 1.309	1.370 1.344 1.321	1.384 1.356 1.333	1.397 1.369 1.344	1.411 1.381 1.356	1.424 1.394 1.368	1.438 1.406 1.379	1.451 1.419 1.391	1.464 1.431 1.402	1.491 1.456 1.426	1.518 1.481 1.449	1.545 1.506 1.472	1.572 1.531 1.496	1.599 1.556 1.519	1.626 1.581 1.542	1.653 1.606 1.566	1.679 1.631 1.589	1.706 1.656 1.612	1.733 1.681 1.635

$$\frac{b_{\mathbf{F}}}{b_{\mathbf{W}}} = 0.4 - \text{Concluded}$$

 $\frac{t_{\overline{w}}}{t_{\overline{S}}} = 1.00$ 

bg/tg	20	21	22	23	24	25	26	27	28	29	30	32	孙	36	38	μο	2با	Įį į	46	148	50
25.67.88 20.7.88 20.7.89 20.7.	2.327 2.276 2.228 2.185 2.144 2.106	2.383 2.330 2.280 2.235 2.192 2.152	2.439 2.383 2.332 2.285 2.240 2.199	2.495 2.437 2.384 2.335 2.289 2.246	2.551 2.491 2.436 2.385 2.337 2.292	2.607 2.545 2.488 2.435 2.385 2.339	2.663 2.599 2.540 2.485 2.433 2.386	2.719 2.653 2.591 2.535 2.482 2.432	2.775 2.706 2.643 2.585 2.530 2.479	2.831 2.760 2.695 2.635 2.578 2.526	2.887 2.814 2.747 2.685 2.626 2.572	2.999 2.922 2.851 2.785 2.723 2.666	3.111 3.030 2.954 2.885 2.820 2.759	3.223 3.137 3.058 2.985 2.916 2.852	3.355 3.245 3.162 3.085 3.013 2.946	3.447 3.353 3.265 3.185 3.109 3.039	3.559 3.460 3.369 3.285 3.206 3.132	3.671 3.568 3.473 3.385 3.302 3.226	3.783 3.676 3.577 3.485 3.399 3.319	3.895 3.783 3.680 3.585 3.495 3.412	4.007 3.891 3.784 3.685 3.592 3.506
51 52 37 34 35	2.070 2.036 2.005 1.975 1.948	2.115 2.080 2.048 2.017 1.988	2.160 2.124 2.090 2.058 2.028	2.205 2.168 2.132 2.099 2.068	2.251 2.211 2.175 2.140 2.108	2.296 2.255 2.217 2.181 2.148	2.341 2.299 2.260 2.223 2.188	2.386 2.343 2.302 2.264 2.228	2.432 2.386 2.344 2.305 2.268	2.476 2.430 2.387 2.346 2.308	2.522 2.474 2.429 2.387 2.348	2.612 2.561 2.514 2.470 2.428	2.702 2.649 2.599 2.552 2.508	2.792 2.736 2.684 2.634 2.588	2.883 2.824 2.769 2.717 2.668	2.973 2.911 2.854 2.799 2.748	3.063 2.999 2.938 2.881 2.828	3.154 3.086 3.023 2.964 2.908	3.244 3.174 3.108 3.046 2.988	3.334 3.261 3.193 3.128 3.068	3.425 3.349 3.278 3.211 3.148
38 39 40	1.873 1.850 1.829	1.954 1.910 1.886 1.864	1.946 1.922 1.899	1.983 1.958 1.954	2.020 1.994 1.969	2.004	2.094 2.066 2.039	2.161 2.131 2.102 2.074	2.168 2.138 2.139	2.237 2.204 2.174 2.144	2.275 2.241 2.209 2.179	2.350 2.315 2.281 2.249	2.426 2.389 2.353 2.319	2.462 2.425 2.425 2.389	2.578 2.536 2.497 2.459	2.653 2.610 2.568 2.529	2.729 2.683 2.640 2.599	2.805 2.757 2.712 2.669	2.880 2.831 2.784 2.739	2.956 2.904 2.856 2.809	3.032 2.978 2.927 2.879
, , ,			-/	11-1	~~,,,			- • • ) /	1100,	1/-/	-•>42	***	16)		2.101	2.22	20013	12.222	E • 27 L	12.447	2.790 2.708 2.634 2.566 2.503
52 556 58 60	1.638 1.614 1.592 1.572 1.553	1.665 1.640 1.617 1.596 1.576	1.692 1.666 1.642 1.620 1.599	1.719 1.692 1.667 1.644 1.623	1.746 1.718 1.692 1.668 1.646	1.772 1.744 1.717 1.693 1.669	1.799 1.770 1.742 1.717 1.693	1.826 1.796 1.767 1.741 1.716	1.853 1.822 1.792 1.765 1.739	1.880 1.848 1.817 1.789 1.763	1.907 1.873 1.842 1.813 1.786	1.961 1.925 1.892 1.861 1.833	2.015 1.977 1.942 1.910 1.879	2.069 2.029 1.992 1.958 1.926	2.122 2.081 2.042 2.006 1.973	2.176 2.133 2.092 2.055 2.020	2.230 2.185 2.142 2.103 2.066	2.284 2.236 2.192 2.151 2.113	2.338 2.288 2.242 2.199 2.160	2.392 2.340 2.292 2.248 2.206	2.446 2.392 2.342 2.296 2.253
65 70 75	1.510 1.474 1.442	1.532 1.494 1.461	1.553 1.514 1.480	1.575 1.534 1.498	1.596 1.554 1.517	1.618 1.574 1.536	1.639 1.594 1.554	1.661 1.614 1.573	1.683 1.634 1.592	1.704 1.654 1.610	1.726 1.674 1.629	1.769 1.714 1.666	1.812 1.754 1.704	1.855 1.794 1.741	1.898 1.834 1.778	1.941 1.874 1.816	1.984 1.914 1.853	2.027 1.954 1.890	2.070 1.994 1.928	2.113 2.034 1.965	2.156 2.074 2.002

TABLE 3

VALUES OF A<sub>1</sub>/t<sub>S</sub> FOR PLAT PANELS WITH Z-SECTION STIPPENERS. 
$$\begin{bmatrix} \frac{b_W}{b_W} = 0.5. \\ \frac{b_W}{t_S} = 1 + \frac{\frac{b_W}{b_W} \left(1 + \frac{b_F}{b_W}\right) + \frac{b_A}{t_W} - \left(2 - \frac{w}{2}\right) \left(\frac{r_A}{t_W} + \frac{r_F}{t_W} + 1\right)}{\frac{b_S}{t_S}} \begin{pmatrix} t_W \\ t_S \end{pmatrix}^2 \end{bmatrix}$$

bg/tg	20	21	22	23	భ	25	26	27	28	29	30	32	犲	36	38	ħо	142	Щ	146	48	50
27 28 29	1.366 1.353 1.340	1.380 1.367 1.354	1.395 1.381 1.367	1.409 1.394 1.381	1.424 1.408 1.394	1.438 1.422 1.408	1.452 1.436 1.421	1.467 1.450 1.435	1.481 1.464 1.448	1.496 1.478 1.462	1.510 1.492 1.475	1.539 1.520 1.502	1.568 1.548 1.529	1.597 1.576 1.556	1.626 1.603 1.583	1.655 1.631 1.610	1.684 1.659 1.636	1.712 1.687 1.663	1.741 1.715 1.690	1.770 1.743 1.717	1.863 1.830 1.799 1.771 1.744 1.719
1 1	1.299 1.299 1.290	1.321 1.311 1.302 1.293	1.333 1.323 1.313 1.304	1.345 1.355 1.325 1.316	1.357 1.347 1.336 1.327	1.370 1.358 1.348 1.338	1.382 1.370 1.359 1.349	1.394 1.382 1.371 1.360	1.406 1.394 1.382 1.371	1.418 1.406 1.394 1.382	1.430 1.417 1.405 1.394	1.455 1.441 1.428 1.416	1.479 1.465 1.451 1.438	1.504 1.488 1.474 1.460	1.528 1.512 1.497 1.483	1.552 1.536 1.520 1.505	1.577 1.559 1.543 1.527	1	1.626 1.607 1.589 1.572	1.650 1.630 1.612 1.594	1.674 1.654 1.635 1.616
37 38	1.267	1.277	1.288	1,298	1.309	1.320	1.330	1.341	1.351	1.362	1.372	1.393	1.414	1.436	1.457	1.478 1.465	1.499	1.534 1.520 1.506 1.493 1.481	1.541 1.527	1.562 1.547	11.5831
1 116	1.215	1.223	1.232	1.240	1.249	1.257	1.266	1.274	1.283	1.291	1.299	1.316	11.333	1.350	1.367	1.384	1.401	11.418	1.435	11.452	1.514 1.490 1.469 1.450 1.432
54 56 58	1.183 1.176 1.170	1.190 1.183 1.177	1.197 1.190 1.184	1.205 1.197	1.212 1.204 1.197	1.219 1.211 1.20h	1.226 1.218 1.211	1.233	1.241 1.232 1.22h	1.248 1.239 1.231	1.255 1.246 1.237	1.270 1.260 1.251	1.284 1.274 1.264	1.298 1.288 1.278	1.302	1.327 1.316 1.305	1.342 1.330 1.318	!1.332	1.371 1.357 1.365	1.385	1.400
1 70	1.141	1.147	1.152	1.158	1.163	1.169	1.175	1.180	1.186	1.191	1.197	1.208	1.219	1.230	1.241	1.253	1.264	1.296 1.275 1.256	1,286	1.297	1.332 1.308 1.288

TABLE 3
$$\frac{b_F}{b_W} = 0.5 - Continued$$

$$\frac{t_W}{t_S} = 0.63$$

bg/tg	20	21	22	23	2나	25	26	27	28	29	30	32	¾	36	38	140	ЦZ	孙	46	48	50
29 30	1.496	1.516	1.554	1.574	1.575	1.595	1.615	1.635	1.654	1.674	1.694	1:73	1.774	1.813	1.853	1.893	1.932	1.972	2.012	2.051	
35	1.425	1.442	1.459	1.476	1.495	1.510	1.251	1.0744	1.701	1.210	4.777	1.029	1.000	1.691	1.121	1.102	10177	1.077	1.00/	1.901	
		•	4			•			1			3		1	1	3	1				1.909 1.885 1.861 1.839 1.818
42 44 46 48 50	1.354 1.338 1.323 1.310 1.297	1.368 1.352 1.336 1.322 1.309	1.382 1.365 1.349 1.335 1.321	1.397 1.379 1.362 1.347 1.333	1.411 1.392 1.375 1.359 1.345	1.425 1.406 1.388 1.372 1.357	1.439 1.419 1.410 1.384 1.369	1.453 1.433 1.414 1.397 1.381	1.468 1.446 1.427 1.409 1.393	1.482 1.460 1.440 1.421 1.405	1.496 1.473 1.453 1.434 1.417	1.524 1.500 1.479 1.459 1.440	1.553 1.527 1.505 1.483 1.464	1.581 1.554 1.530 1.508 1.488	1.609 1.582 1.556 1.533 1.512	1.638 1.609 1.582 1.558 1.536	1.666 1.636 1.608 1.583 1.559	1.694 1.663 1.634 1.608 1.583	1.723 1.690 1.660 1.632 1.607	1.751 1.717 1.686 1.657 1.631	1.779 1.744 1.712 1.682 1.655
l él.	13 275	וז ספג	11 207	in zog	11 220	11 221	11 まれつ	1 353	13. 361.	11. 375	11.386	Boil. FI	11.カスの	11.452	11.76	11.1.06	11.518	11.500	11.562	In col.	1.629 1.606 1.585 1.564 1.546
65 70 75	1.229 1.212 1.198	1.238 1.221 1.206	1.247 1.229 1.214	1.256 1.238 1.222	1.265 1.246 1.230	1.275 1.255 1.238	1.284 1.263 1.246	1.293 1.272 1.254	1.302 1.281 1.262	1.311 1.289 1.270	1.320 1.298 1.278	1.339 1.315 1.294	1.357 1.332 1.309	1.375 1.349 1.325	1.394 1.368 1.341	1.412 1.383 1.357	1.430 1.400 1.373	1.449 1.417 1.389	1.467 1.434 1.405	1.485 1.451 1.421	1.504 1.468 1.436

TABLE 3

$$\frac{b_{\mathbf{p}}}{b_{\mathbf{W}}} = 0.5 - Continued$$

$$\frac{\mathbf{t_{\overline{W}}}}{\mathbf{t_{\overline{S}}}} = 0.79$$

by/ty	20	21	22	23	21;	25	26	27	28	29	30	32	独	36	38	цо	عبا	孙	46	148	50
25 26 27 28 29 30	1. (0)	1.017	1 T * OH 7	17.074	11.412	1 1 400	1.476	12.009	V . UILL	12.075	ノーエロン	12.170	12.235	12.299	レンころわル!	レン・ハンド	レン・ルロス	12.557	1つ トンコ	12.000	3.031 2.953 2.881 2.814 2.751 2.693
31 32 33 34 35	1.732 1.709 1.688 1.668 1.648	1.762 1.738 1.716 1.695 1.675	1.793 1.768 1.744 1.723 1.702	1.823 1.797 1.773 1.750 1.729	1.853 1.826 1.801 1.778 1.755	1.883 1.856 1.830 1.805 1.782	1.913 1.885 1.858 1.833 1.809	1.943 1.914 1.886 1.860 1.836	1.974 1.943 1.915 1.888 1.862	2.004 1.973 1.943 1.915 1.889	2.034 2.002 1.971 1.943 1.916	2.095 2.060 2.028 1.998 1.969	2.155 2.119 2.085 2.053 2.023	2.215 2.177 2.142 2.108 2.076	2.276 2.236 2.198 2.163 2.130	2.336 2.294 2.255 2.218 2.183	2.396 2.353 2.312 2.273 2.237	2.457 2.411 2.369 2.328 2.290	2.517 2.470 2.425 2.344 2.344	2.578 2.528 2.482 2.438 2.397	2.638 2.587 2.539 2.494 2.451
38 39 40	1.613 1.597 1.582 1.567	1.639 1.622 1.606 1.591	1.664 1.647 1.630 1.614	1.689 1.671 1.554 1.638	1.715 1.696 1.678 1.661	1.740 1.720 1.702 1.684	1.765 1.745 1.726 1.708	1.790 1.770 1.750 1.731	1.816 1.794 1.774 1.755	1.841 1.819 1.798 1.778	1.866 1.844 1.822 1.801	1.917 1.893 1.870 1.848	1.968 1.942 1.918 1.895	2.018 1.991 1.966 1.942	2.069 2.041 2.014 1.989	2.119 2.090 2.062 2.035	2.170 2.139 2.110 2.082	2.221 2.189 2.158 2.129	2.271 2.238 2.206 2.176	2.322 2.287 2.254 2.223	2.411 2.372 2.336 2.302 2.270
42 44 46 48 50	1.540 1.516 1.493 1.473 1.454	1.563 1.537 1.514 1.492 1.473	1.585 1.558 1.534 1.512 1.491	1.607 1.580 1.554 1.531 1.510	1.630 1.601 1.575 1.551 1.529	1.652 1.622 1.595 1.570 1.548	1.674 1.643 1.615 1.590 1.566	1.696 1.665 1.636 1.609 1.585	1.719 1.686 1.656 1.629 1.604	1.741 1.707 1.677 1.648 1.622	1.763 1.729 1.697 1.668 1.641	1.808 1.771 1.738 1.707 1.679	1.852 1.814 1.778 1.746 1.716	1.897 1.856 1.819 1.785 1.753	1.942 1.899 1.860 1.824 1.791	1.986 1.941 1.900 1.863 1.828	2.031 1.984 1.941 1.902 1.866	2.075 2.026 1.982 1.941 1.903	2.120 2.069 2.023 1.980 1.941	2.165 2.112 2.063 2.019 1.978	2.209 2.154 2.104 2.058 2.016
1 5h	11.11.20	11.1.28	11.455	17.1172	ימפיו. דו	11.5071	1.526	11.5/12/	1.550	11.576	1.50	11.6281	11.663	ואסא נו	1 732	1 767	เา ผกว	11 BX6	11 271	וז ההצ	1.977 1.940 1.907 1.875 1.846
65 70 75	1.349 1.324 1.303	1.364 1.338 1.315	1.378 1.351 1.328	1.392 1.364 1.340	1.407 1.378 1.353	1.421 1.391 1.365	1.436 1.404 1.377	1.450 1.418 1.390	1.464 1.431 1.402	1.479 1.445 1.415	1.493 1.458 1.427	1.522 1.485 1.452	1.551 1.511 1.477	1.580 1.538 1.502	1.608 1.565 1.527	1.637 1.592 1.552	1.666 1.618 1.577	1.695 1.645 1.602	1.724 1.672 1.627	1.752 1.699 1.652	1.781 1.725 1.677

TABLE 3

$$\frac{b_{\mathbf{F}}}{b_{\mathbf{W}}} = 0.5 - \text{Concluded}$$

$$\frac{t_{\overline{W}}}{t_{\overline{Q}}} = 1.00$$

bs/ts	20	21	22	23	SĮţ	25	26	27	28	29	30	32	34	36	38	40	42	抐	146	148	50
25 26 27 28 29 30	2.407 2.353 2.302 2.256 2.213 2.172	2.467 2.410 2.358 2.310 2.264 2.22	2.527 2.468 2.414 2.363 2.316 2.272	2.587 2.526 2.469 2.417 2.368 2.322	2.647 2.583 2.525 2.470 2.420 2.372	2.707 2.641 2.580 2.524 2.471 2.422	2.767 2.699 2.636 2.577 2.523 2.472	2.827 2.756 2.691 2.631 2.575 2.522	2.887 2.814 2.747 2.685 2.626 2.572	2.947 2.872 2.802 2.738 2.678 2.622	3.007 2.930 2.858 2.792 2.730 2.672	3.127 3.045 2.969 2.899 2.833 2.772	3.247 3.160 3.080 3.006 2.937 2.872	3.367 3.276 3.191 3.113 3.040 2.972	3.487 3.391 3.302 3.220 3.144 3.072	3.607 3.506 3.414 3.327 3.247 3.172	3.727 3.622 3.525 3.435 3.351 3.272	3.847 3.737 3.636 3.542 3.454 3.372	3.967 3.853 3.747 3.558 3.472	4.087 3.968 3.858 3.756 3.661 3.572	4.207 3.083 3.969 3.863 3.764 3.672
1	ţ	j	2.231 2.193 2.157 2.123 2.090	) i			J		J	1		1	3		<b>S</b>	}	1	1	1	]	3.586 3.505 3.429 3.358 3.290
37 38 39 40	1.950 1.925 1.902 1.879	1.991 1.965 1.940 1.917	1.979	2.072 2.044 2.017 1.992	2.113 2.083 2.056 2.029	2.153 2.123 2.094 2.067	2.194 2.162 2.133 2.104	2.234 2.202 2.171 2.142	2.275 2.241 2.209 2.179	2.315 2.281 2.248 2.217	2.356 2.320 2.286 2.254	2.437 2.399 2.363 2.329	2.518 2.478 2.440 2.404	2.599 2.557 2.517 2.479	2.636 2.594 2.554	2.761 2.715 2.671 2.629	2.794 2.748 2.704	2.923 2.873 2.825 2.779	3.005 2.952 2.902 2.854	3.086 3.031 2.979 2.929	3.227 5.167 3.110 3.056 3.004
42 44 48 50	1.837 1.799 1.764 1.733 1.703	1.873 1.833 1.797 1.764 1.733	1.909 1.867 1.830 1.795 1.763	1.944 1.901 1.862 1.826 1.793	1.980 1.936 1.895 1.858 1.823	2.016 1.970 1.928 1.889 1.853	2.052 2.004 1.960 1.920 1.883	2.087 2.038 1.993 1.951 1.913	2.123 2.072 2.025 1.983 1.943	2.159 2.106 2.058 2.014 1.973	2.195 2.140 2.091 2.045 2.003	2.266 2.208 2.156 2.108 2.063	2.337 2.277 2.221 2.170 2.123	2.409 2.345 2.286 2.233 2.183	2.480 2.413 2.351 2.295 2.243	2.552 2.481 2.417 2.358 2.303	2.623 2.549 2.482 2.420 2.363	2.695 2.617 2.547 2.485 2.423	2.766 2.686 2.612 2.545 2.483	2.837 2.754 2.678 2.608 2.543	2.909 2.822 2.743 2.670 2.603
52 51 58 50	1.651 1.628	1.679	1.682 1.658	1.735 1.708	1.762 1.735	1.790 1.762 1.736	1.818 1.789	1.846 1.815 1.787	1.873 1.842 1.813	1.901 1.869 1.839	1.929 1.896	1.985 1.949	2.040 2.003	2.096 2.057 2.020	2.151 2.110 2.072	2.207	2.262	2.318 2.271 2.227	2.374	2.429	2.542 2.485 2.432 2.381 2.336
65 70 75	1.541 1.502 1.469	1.564 1.524 1.489	1.587 1.545 1.509	1.610 1.567 1.529	1.633 1.588 1.549	1.656 1.610 1.569	1.679 1.631 1.589	1.703 1.652 1.609	1.726 1.674 1.629	1.749 1.695 1.649	1.772 1.717 1.669	1.818 1.760 1.709	1.864 1.802 1.749	1.910 1.845 1.789	1.956 1.888 1.829	2.003 1.931 1.869	2.049 1.974 1.909	2.095 2.017 1.949	2.141 2.060 1.989	2.187 2.102 2.029	2.233 2.145 2.069

TABLE 4

VALUES AND COMPUTATIONS FOR OBTAINING IDEAL DESIGN

[Pi =	3.0	kips/in.;	c =	ij
-------	-----	-----------	-----	----

	Step 1		Ster	2		Step 3	Step 4		Step 5	
L (in.)	$\frac{\frac{P_{1}}{L/\sqrt{c}}}{\frac{(\text{kips/in.})}{\text{in.}}}$	tw to	b <u>s</u>	b <sub>₩</sub>	σ <sub>f</sub>	A <sub>1</sub>	t <sub>S</sub>	t <sub>W</sub>	b <sub>S</sub>	b <sub>W</sub>
10	0.30	0.51 .63 .79 1.00	27 28 29 29	26 25 24 24	a34.0 35.6 36.7 37.4	1.427 1.602 1.860 2.337	*0.0618 .0526 .0440 .0343	*0.0315 .0331 .0348 .0343	*1.67 1.47 1.28 1.00	a <sub>0.82</sub> .83 .83 .82
20	.15	.51 .63 .79 1.00	32 33 34 35	32 31 29 28	28.7 <b>a</b> 30.4 31.6 32.2	1.429 1.612 1.862 2.268	.0732 a.0612 .0510 .0411	.0373 a.0386 .0403 .0411	2.34 <b>2</b> .02 1.73 1.44	1.19 a1.20 1.17 1.15
30	.10	.51 .63 .79 1.00	34 35 37 38	37 35 33 31	25.0 827.1 27.8 28.6	1.457 1.640 1.886 2.278	.0824 •.0675 .0572 .0461	.0421 a.0425 .0452 .0461	2.80 <b>a</b> 2.36 2.12 1.75	1.56 a1.49 1.49 1.43

avalues indicating designs that approach requirement of  $t_8$  = 0.064 in.

TABLE 5

VALUES AND COMPUTATIONS FOR OBTAINING PRACTICAL DESIGN BY SHORT METHOD  $\begin{bmatrix} P_1 = 3.0 \text{ kips/in.;} & L = 20 \text{ in.;} & c = 1; & t_S = 0.06 \text{L} & in.; & \frac{t_W}{t_S} = 0.79 \end{bmatrix}$ 

Step 1	Step 1 Step 2		Step 3	Step 4		Step	5	St	tep 6	Step 7				
$\begin{pmatrix} \frac{P_1}{L/\sqrt{c}} \\ \left(\frac{\text{kips/in.}}{\text{in.}}\right) \end{pmatrix}$	bs Es	b <sub>W</sub>	σ <sub>f</sub> (ksi)	A <sub>1</sub>	tg	bg tg	bw tw s = 0.	σ <sub>f</sub> (ksi)	te	P <sub>i</sub> kips/in.	tw (in.)	bs (in.)	bw (in.)	or (ks1)
	30 35 40 50	30 30 28 26	30.9 31.7 29.7 27.1	2.006 1.862 1.711 1.534	0.0484 .0508 .0590 .0722		26.1	28.8	1.619	2.98	0.051		1.33	23.5

TABLE 6

VALUES AND COMPUTATIONS FOR OBTAINING DESIGN FOR MAXIMUM STRUCTURAL EFFICIENCY  $\begin{bmatrix} P_1 = 3.0 \text{ kips/in.;} & L = 20 \text{ in.;} & c = 1; & t_S = 0.06 \text{µ in.;} & \frac{t_W}{t_S} = 0.79 \end{bmatrix}$ 

Step 1	:	Step	2	Step 3	Step 4 Step 7			St	ep 8		Step	9		
$\begin{pmatrix} \frac{P_{1}}{L/\sqrt{c}} \\ \frac{(\text{kips/in.})}{\text{in.}} \end{pmatrix}$	b <sub>W</sub>	bs ts	$\overline{\sigma}_{\mathbf{f}}$ (ksi)	A <sub>1</sub>	t <sub>S</sub>	bs ts For t	bw tw s = 0.0	(ksi)	t <sub>S</sub>	P <sub>i</sub> kips/in.	t <sub>w</sub>	b <sub>S</sub>	b <sub>W</sub>	Gr (ksi)
0.15	20	25 <b>30</b> <b>35</b> 40	26.4 27.1 27.3 27.8	1.858 1.715 1.613 1.536	0.0612 .0646 .0682 .0702	42.1	25.0	29.0	1.612	2.99	0.051	2.69	1.27	24.6
	25	30 35 40 50	29.9 30.2 29.5 27.1	1.861 1.738 1.645 1.516	.0549 .0572 .0618 .0730									
	30	35 40 50	31.7 29.6 26.9 24.5	1.862 1.755 1.604 1.503	.0508 .0578 .0696 .08 <b>1</b> 4									
	40	35 40 50	26.2 25.8 23.8 22.6	2.112 1.973 1.778 1.649	.0542 .0589 .0709 .0805									
	50	35 45 50 50	23.2 23.4 22.3 20.7	2.362 2.192 1.953 1.794	.0547 .0584 .0688 .0812									

TABLE 7

TEST DATA ON WHICH DESIGN CHARTS ARE BASED FOR 24s-T ALUMINUM-ALLOY FLAT PANELS WITH LONGITUDINAL Z-SECTION STIFFENERS

= 0.51

									<del>t</del> s =	0.51									
bs ts	by ty	b <sub>F</sub>	σ̄ <sub>f</sub>	Fi L/vo (kips/in.) in.	\$   ts	by tw	p# ph	σ <sub>f</sub> (k•1)	$ \frac{\frac{F_1}{L/\sqrt{6}}}{\frac{kips/in.}{in.}} $	bs ts	by ty	p <sup>#</sup>	σ <sub>f</sub>	L/ve (kipe/in.)	bs ts	by €	b <u>r</u>	ਰੋ <sub>ਵ</sub> (kei)	Fi L/vc (kips/in.)
25	20	0.4	37.0 33.7 25.2 14.8	0.646 •365 •188 •082	35,	25	0.5	32.3 31.4 27.0 14.4	0.486 .271 .163 .062	50	25	0.4	25.9 24.0 22.2 13.6	0.431 .217 .144 .061	75	25	0.3	21.0 18.6 16.5 13.7	0.378 .191 .118 .069
	25	-4	36.3 33.8 26.0 15.6	•550 •298 •159 •066		30	•3	31.4 30.7 27.0 16.8	.427 .235 .145 .064			•5	26.5 24.9 22.8 15.9	.403 .217 .140 .069			۰Ħ	21.0 19.4 17.5 12.2	.360 .189 .121 .059
	30	.4	33.8 31.3 26.0 15.4	.142 .227 .131 .056			-4	30.7 31.0 27.4 17.4	.391 .227 .141 .064		30	.3	25.6 24.3 22.6 15.7	•355 •192 •123 •061			•5	20.9 19.4 17.6 12.8	.34.5 .182 .115 .061
	<b>4</b> 0	<b>.</b> 4	29.3 23.9 22.5 16.8	.295 .137 .090 .048			•5	31.1 30.3 27.5 17.5	.392 .242 .138 .062			-ф	25.0 25.8 22.6 16.4	.323 .176 .117 .060		30	•3	19.9 18.8 16.4 13.6	.289 .155 .094 .055
	50	•4	25.9 20.5 17.7 16.4	.222 .101 .061 .049		<b>4</b> 0	-lt	26.5 25.6 23.6 16.8	.252 .142 .091 .046			•5	25.5 24.4 23.0 16.5	.321 .174 .116 .059			.4	19.1 19.8 17.7 13.4	.268 .156 .098 .053
35	20	•3	31.5 30.6 25.6 14.9	.6    .358 .211 .088		50	•4	23.9 21.8 20.8 16.0	.183 .097 .066 .042		40	•14	23.3 22.2 20.2 15.7	.219 .118 .078 .041			•5	20.6 19.8 17.0 13.2	.272 .1148 .089 .049
		•4	31.2 31.2 25.8 15.2	.611 .347 .203 .086	50	20	•5	26.2 24.0 21.9 15.0	•577 •300 •193 •092		50	•4	21.3 19.1 18.0 15.6	.159 .080 .054 .037		140	4	19.2 17.9 16.8 12.8	.186 .100 .065 .035
		•5	31.4 30.3 26.4 13.4	.587 .325 .202 .073			<b>.</b> lı	26.7 25.1 21.2 14.6	•557 •297 •177 •087	75	20	•3	22.4 18.4 16.5 12.2	•591 •252 •158 •084		50	•jt	17.0 16.2 14.9 11.7	.129 .070 .046 .025
	25	-3	32.5 31.7 26.6 16.8	•535 •295 •171 •078			•5	26.3 23.7 21.1 15.0	.531 .266 .170 .086			• <b>†</b>	20.5 19.2 16.7 12.0	.471 .245 .149 .078					

.430 .241 .140 .070

25.2 24.7 20.8 15.2

30.9 31.2 26.9 13.4 .477 .278 .169 25

TABLE 7
TEST DATA - Continued

 $\frac{t_W}{t_S} = 0.65$ 

									٠8										
<u> Եց</u>	b₩ tw	₩ बंद	ξ <sub>f</sub> (ksi)	$\frac{\frac{P_1}{L/\sqrt{c}}}{\frac{\text{kips/in.}}{\text{in.}}}$	bg tg	#   #	p <sup>M</sup>	σ <sub>f</sub> (ka1)	r <sub>1</sub> L/vc (rips/in.) in.	bg tg	#   E	b₽ D₩	σ <sub>f</sub>	\frac{\fir}}}}}}}{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\fir}}}}}}{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\frac{\fin}}}}}{\frac{\frac{\frac{\frac{\frac{\frac{\fir}}}}}{\frac	bg ts	b₩ ŧ₩	b <sub>F</sub>	σ̄ <sub>f</sub>	P <sub>1</sub> L/\sqrt{a} \left(\frac{\kips/in.}{in.}\right)
25	20	0.4	38.9 53.4 24.9 14.3	0.626 .307 .159 .065	<b>3</b> 5	25	0.5	32.5 30.9 28.2 17.4	0.429 .238 .158 .068	50	25	0.1	28.0 25.0 20.4 13.5	0.323 .168 .094 .044	75	25	0.3	22.7 19.2 15.6 12.1	0.281 .134 .076 .043
	25	•4	38.9 32.4 24.8 14.2	.517 .242 .132 .054		30	•3	30.7 29.9 27.9 17.9	•335 •186 •122 •055			•5	28.0 26.2 23.0 17.2	.365 .183 .117 .063			-4	22.1 19.8 16.4 11.3	.257 .132 .076 .038
	30	-4	36.3 30.9 24.6 16.6	.420 .199 .113 .055			•4	30.7 31.4 28.1 18.5	.329 .193 .121 .057		30	-3	26.0 24.9 22.9 16.3	.279 .152 .097 .049			•5	22.7 20.9 18.1 13.8	.304 .161 .097 .053
	40	•4	29.4 24.2 23.5 15.6	.275 .131 .088 .042			.5	30.4 29.1 27.6 18.0	.314 .173 .115 .054			•4	25.8 26.9 23.2 16.2	.266 .167 .095 .048		30	-3	20.6 19.6 18.6 13.8	.226 .121 .081 .043
	50	-4	25.2 22.1 18.7	.210 .104 .061		μo	-4	27.8 24.9 23.8 16.9	.235 .118 .080 .042			•5	26.2 24.0 22.3 17.1	.261 .135 .088 .049			-4	21.2 21.0 18.8 14.1	.219 .128 .079 .042
35	20	.3	32.0 32.8 25.4 15.6	.516 .304 .168 .072		50	-4	24.0 21.9 19.2	.173 .089 .054		40	-4	23.8 22.4 21.4 17.4	.187 .101 .067 .040			•5	21.0 20.2 18.0 14.2	.207 .121 .075 .040
		•4	33.5 34.0 28.3 15.4	•52lı •306 •179 •069	50	20	-3	28.2 25.0 21.8 15.1	.459 .233 .144 .071		50	-4	21.5 19.6 17.9 16.9	.141 .071 .046 .037		40	•#	18.6 18.2 17.5 13.4	.138 .081 .053 .029
		.5	33.5 32.0 29.4 17.7	.506 .275 .178 .075			-14	28.3 25.8 22.2 14.5	.447 .230 .140 .065	75	20	.3	22.0 20.2 15.8 13.0	.381 .206 .110 .065		50	-4	17.7 17.5 16.0 13.1	.105 .061 .039 .024
	25	-3	33.1 32.4 24.7 14.4	.430 .243 .127 .053			.5	27.4 25.5 22.9 16.3	.410 .216 .137 .069			•pt	22.7 21.3 17.2 12.5	.374 .200 .113 .059					

22.4 20.5 18.5 13.9

.419 .235 .130 .055

25

27.4 25.0 20.6 13.5 .329 .171 .099 .047

33.8 32.4 26.1 15.1

TABLE 7
TEST DATA - Continued

 $\frac{t_W}{t_S} = 0.79$ 

							ts							
b <sub>5</sub>	b₩ t₩	p.M.	σ <sub>f</sub>	Ei L/vo (kips/in.)	b <sub>S</sub>	b₩ t₩	p.M. p.M.	$\overline{\sigma}_{\mathbf{f}}$ (ksi)	ri L/√c (kips/in.)	bg ts	PM PM	p.	σ̄ <sub>f</sub>	$\frac{\frac{P_1}{L/\sqrt{c}}}{\frac{kips/in.}{in.}}$
35	20	0.3	34.5 34.4 26.9 16.5	0.470 .270 .148 .063	50	20	0.3	28.7 27.2 23.4 16.2	0.369 .205 .123 .061	75	20	0.3	22.9 21.7 17.4 12.7	0.306 .167 .092 .049
		•14	35.7 34.6 29.7 16.8	.472 .265 .157 .062			<b>.</b> 4	29.6 27.2 23.2 16.4	•374 •194 •115 •059			•l‡	24.7 23.1 20.4 14.2	.313 .168 .106 .052
		. •5	34.8 33.7 27.7 17.6	•149 •249 •141 •065			•5	29.7 28.5 23.1 16.4	.365 .201 .112 .082			•5	24.2 22.3 19.4 13.3	.298 .157 .094 .046
	25	•3	37.0 32.6 28.2 17.6	•394 •202 •126 •056		25	•3	28.9 28.2 24.4 16.9	.295 .165 .100 .049		25	•3	22.2 19.3 13.8	.125 .077 .040
		•14	35.2 33.3 29.3 17.1	•374 •204 •123 •052			•#	29.2 27.8 23.3 17.0	.277 .158 .092 .048		: :	-4	24.4 22.6 19.5 13.4	.236 .125 .075 .037
		•5	33.2 33.8 17.7	•357 •213 •056			•5	28.5 27.4 23.5 17.2	.309 .169 .102 .053			•5	23.6 22.9 19.9 14.2	.242 .137 .083 .041
	30	•3	33.8 32.0 28.5 18.6	•340 •181 •110 •052		30	•3	27.8 26.6 23.6 17.6	.250 .137 .08பு .0ப்பு		30	•3	22.9 22.3 19.2 14.4	.199 .109 .066 .035
		•4	32.5 31.7 28.2 17.5	.313 .173 .107 .048			•14	27.6 26.8 22.6 17.2	.238 .134 .079 .043			•14	23.6 21.6 19.7 14.0	.198 .102 .067 .033
		•5	31.2 30.2 26.5 17.1	.302 .162 .101 .046			•5	26.9 25.0 23.7 16.5	.234 .122 .084 .040			•5	22.5 20.9 18.9 14.8	.182 .095 .061 .037

TABLE 7
TEST DATA - Concluded

 $\frac{\mathbf{t_{W}}}{\mathbf{t_{S}}} = 1.00$ 

							<sup>1</sup> 8							
bg tg	b₩ t₩	. वित्	σ̄ <sub>f</sub> (ks1)	$\frac{\frac{P_1}{L/\sqrt{c}}}{\frac{(\text{kips/in.})}{\text{in.}}}$	<sup>b</sup> 3 t <sub>3</sub>	ç <sup>M</sup> ρ <sup>M</sup>	ρ <b>λ</b> γλ	σ̄ <sub>f</sub> (ks1)	$\begin{pmatrix} \frac{P_1}{L/\sqrt{c}} \\ \frac{\text{kips/in.}}{\text{in.}} \end{pmatrix}$	bg tg	b <sub>₩</sub>	p <sup>™</sup>	σ̄ <sub>f</sub> (ksi)	$\frac{\frac{P_1}{L/\sqrt{o}}}{\frac{(\text{kips/in.})}{\text{in.}}}$
<b>3</b> 5	20	0.3	36.3 32.8 26.8 17.6	0.430 .215 .125 .059	50	20	0.3	30.2 27.8 23.2 18.5	0.320 .165 .096 .051	75	20	0.3	24.0 23.2 18.3 14.6	0,235 .129 .072 .041
	İ	•4	36.7 33.7 27.3 17.5	.418 .222 .122 .057			-4	31.6 29.1 24.1 16.7	.322 .167 .096 .048			•4	25.9 23.8 20.1 14.3	.245 .128 .076 .039
		•5	36.1 34.1 28.0 17.2	.414 .223 .130 .057		,	•5	32.0 28.2 23.2 16.5	.326 .164 .092 .049			•5	26.1 24.7 20.2 14.8	.250 .129 .077 .041
	25	•3	33.5 31.7 27.3 18.1	•333 •180 •108 •052		25	-3	29.6 27.1 24.3 17.2	.260 .134 .084 .042		25	-3	24.2 22.4 20.7 14.7	.200 .101 .066 .033
		•4	35.4 31.8 28.7 16.4	•347 •177 •112 •047			-4	31.0 27.7 24.5 18.0	.262 .138 .083 .046			-4	25.8 24.1 21.1 14.3	.199 .106 .065 .033
		•5	35.3 33.1 25.3 16.8	.358 .190 .103 .048			•5	30.8 27.9 23.9 17.5	.270 .139 .084 .044			•5	25.5 23.9 21.5 14.1	.199 .108 .067 .032
	30	•3	33.1 29.4 23.6 15.0	.290 .149 .084 .037		30	•3	28.2 25.9 23.3 16.6	.213 .112 .070 .037		30	.3	23.9 21.6 19.6 14.5	.161 .084 .053 .028
		<b>.</b> 4	33.3 31.6 23.8 15.1	.291 .163 .083 .038			-4	29.0 27.5 21.8 16.6	.218 .120 .065 .036			•4	24.2 22.8 20.8 14.8	.160 .086 .056 .028
		•5	32.7 30.1 26.0 13.5	.300 .160 .095 .036			•5	28.1 26.3 22.7 16.1	.216 .117 .070 .036			•5	24.0 22.4 20.2 14.1	.163 .086 .055 .027

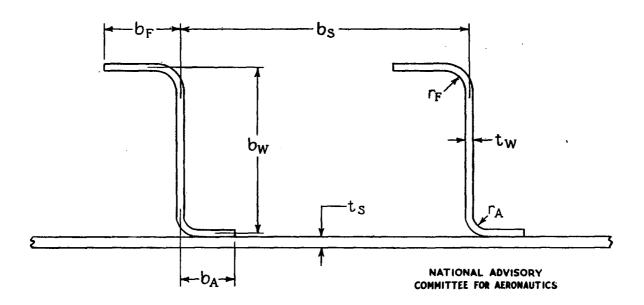


Figure I. - Symbols for panel dimensions.

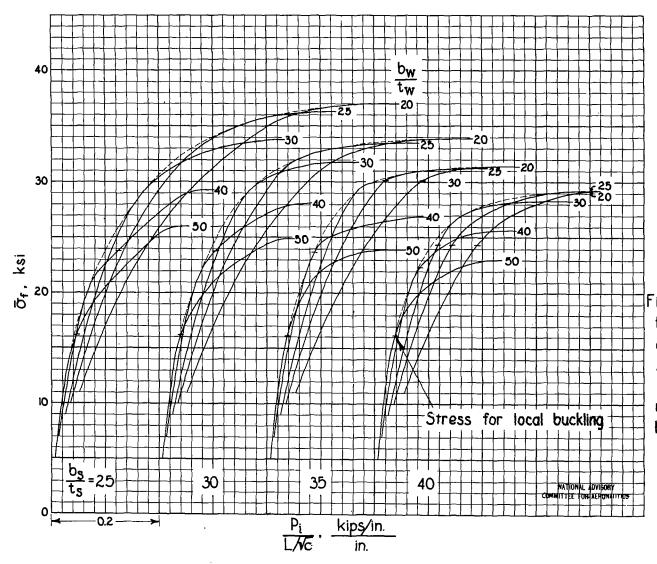


Figure 2.-Design chart for 24 S-T aluminum-alloy flat panels with Z-section stiffeners; tw/t\_s=0.51.(b\_A/t\_w=11.4; r\_A/t\_w=3; r\_F/t\_w=4; and b\_F/b\_w=0.3 to 0.5)

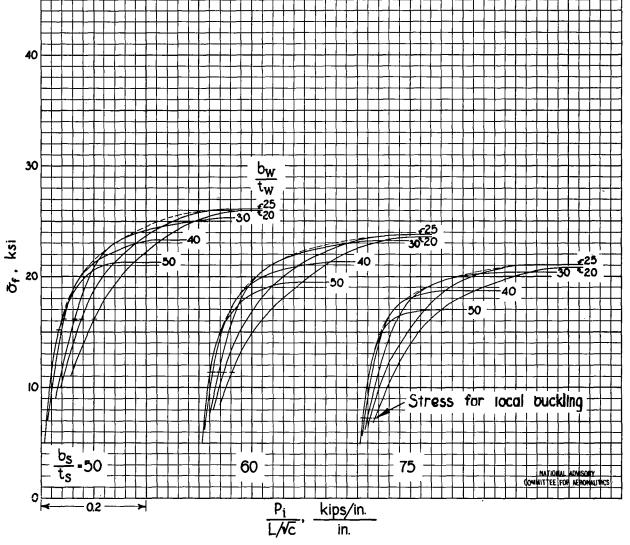


Figure 2.-Concluded.

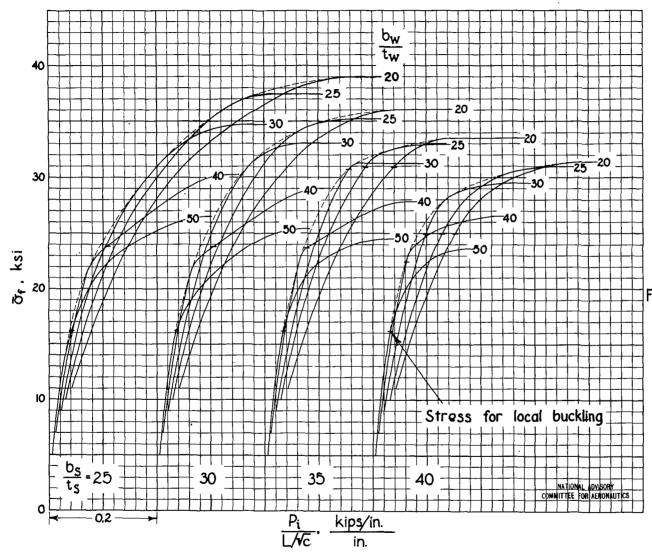


Figure 3.-Design chart for 24S-T aluminum-alloy flat panels with Z-section stiffeners;  $t_{\rm W}/t_{\rm S}$ =0.63.( $b_{\rm A}/t_{\rm W}$ =10.9;  $r_{\rm A}/t_{\rm W}$ =3;  $r_{\rm F}/t_{\rm W}$ =4; and  $b_{\rm F}/b_{\rm W}$ =0.3 to 0.5)

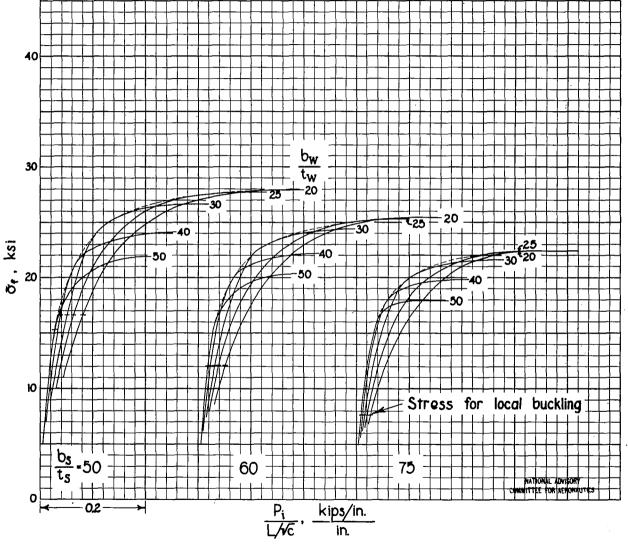


Figure 3.-Concluded.

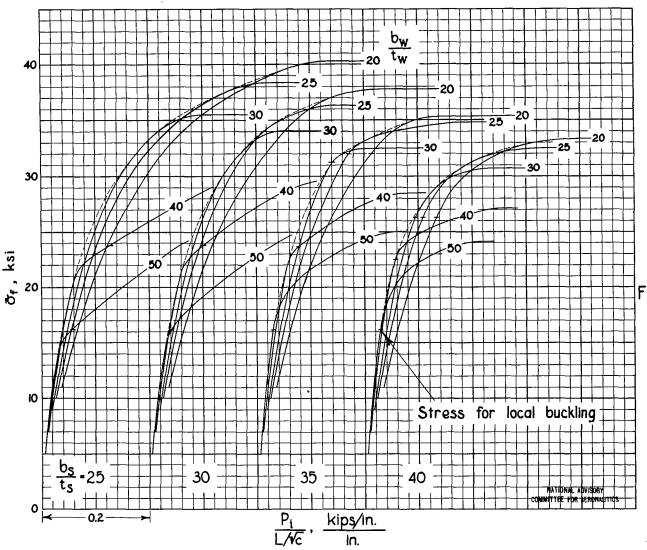


Figure 4.- Design chart for 24S-T aluminum-alloy flat panels with Z-section stiffeners; tw/t<sub>3</sub>=0.79.(b<sub>A</sub>/tw=9.8; r<sub>A</sub>/tw=3; r<sub>F</sub>/tw=4; and b<sub>F</sub>/b<sub>W</sub>=0.3 to 0.5)

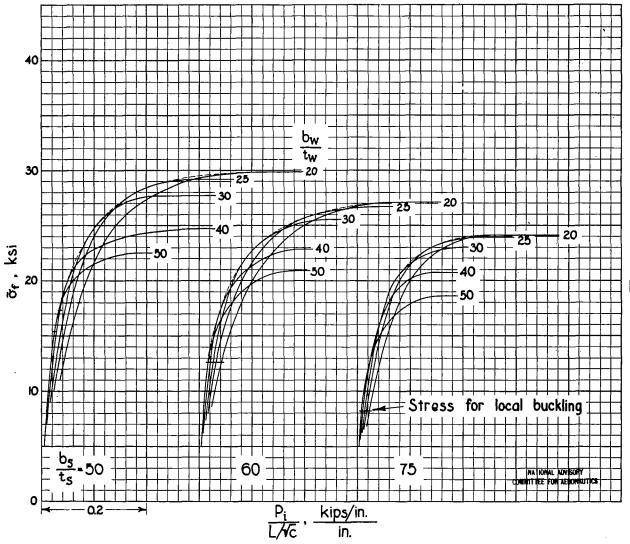


Figure 4.-Concluded.

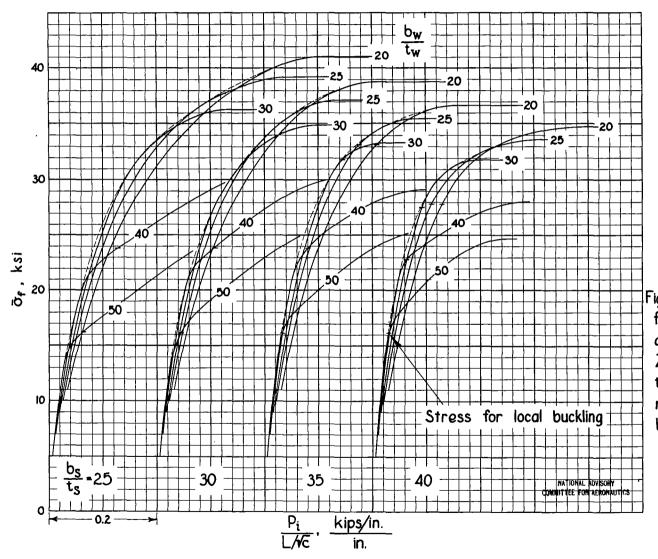


Figure 5.- Design chart for 24S-T aluminum-alloy flat panels with Z-section stiffeners; tw/t<sub>3</sub>=1.00.(b<sub>A</sub>/t<sub>w</sub>=8.6; r<sub>A</sub>/t<sub>w</sub>=3; r<sub>F</sub>/t<sub>w</sub>=4; and b<sub>F</sub>/b<sub>w</sub>=0.3 to 0.5)

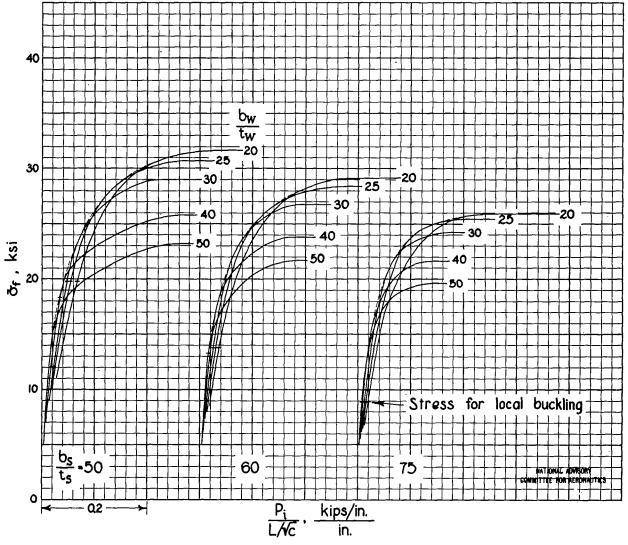


Figure 5.-Concluded.

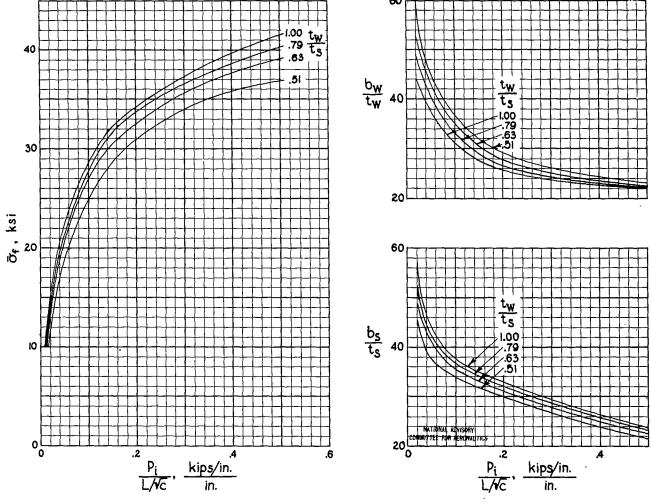


Figure 6. – Highest values of average stress at failure for 24S-T aluminum-alloy flat panels with Z-section stiffeners, with values of  $b_{\rm S}/t_{\rm S}$  and  $b_{\rm W}/t_{\rm W}$  needed to realize these stresses.

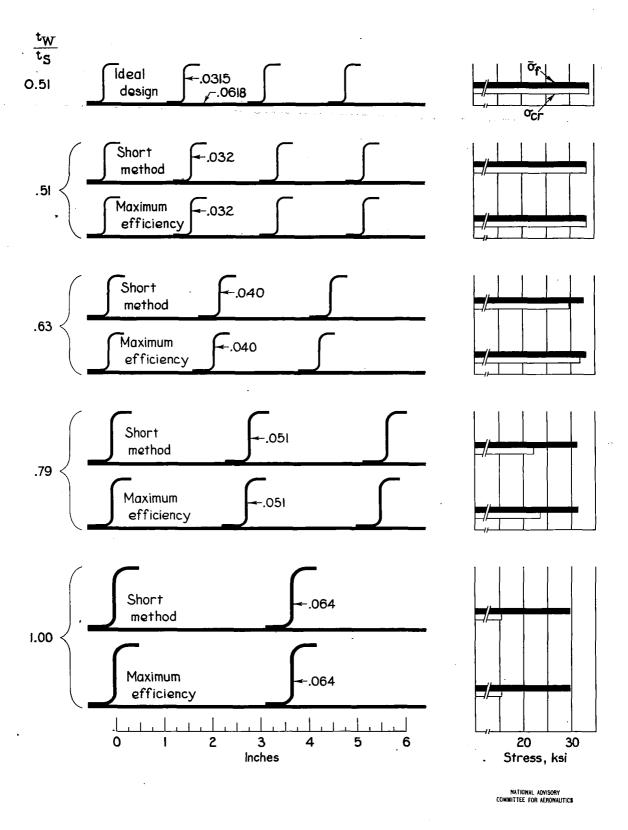


Figure 7.- Designs of 24S-T aluminum-alloy panels 10 inches long with  $P_i$ =3.0 kips/inch, c=1, and  $t_s$ =0.064 inch.

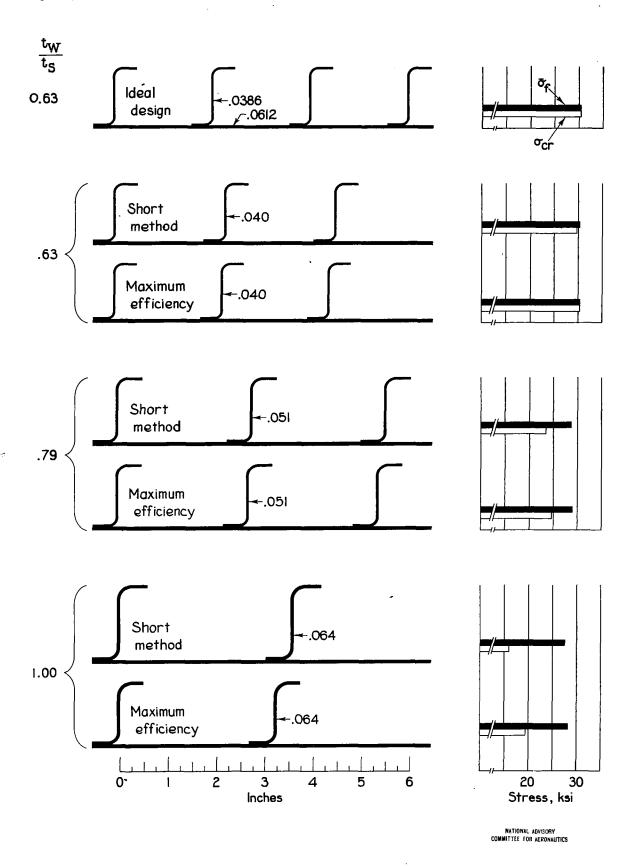


Figure 8.- Designs of 245-T aluminum-alloy panels 20 inches long with  $P_i$ =3.0 kips/inch, c=1, and  $t_s$ =0.064 inch.

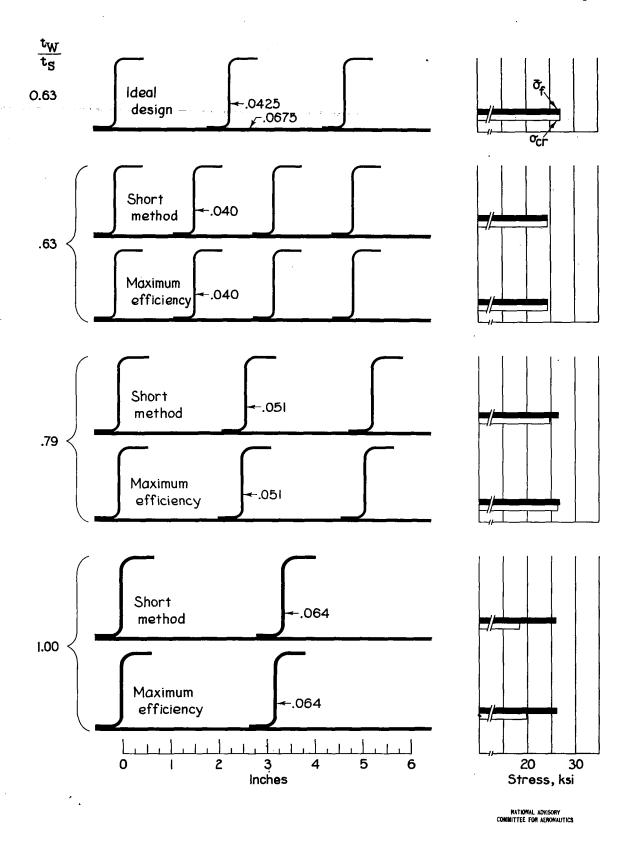


Figure 9.- Designs of 245-T aluminum-alloy panels 30 inches long with  $P_i$ =3.0 kips/inch, c=1, and t<sub>s</sub>=0.064 inch.

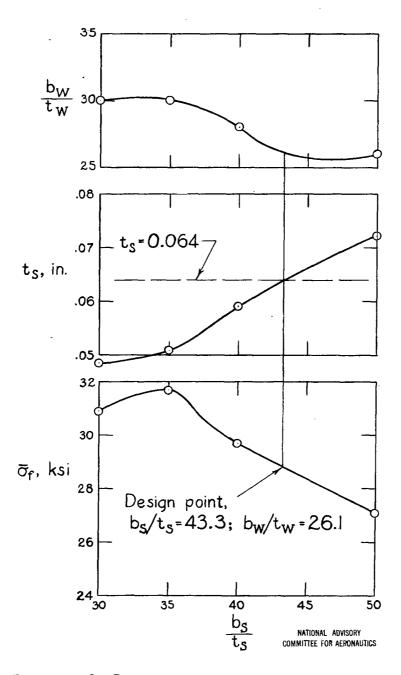


Figure 10.-Plot for obtaining practical design by short method.  $P_i$ =3.0 kips/inch; L=20 inches; c=1; t<sub>s</sub>=0.064 inch; t<sub>w</sub>/t<sub>s</sub>=0.79.

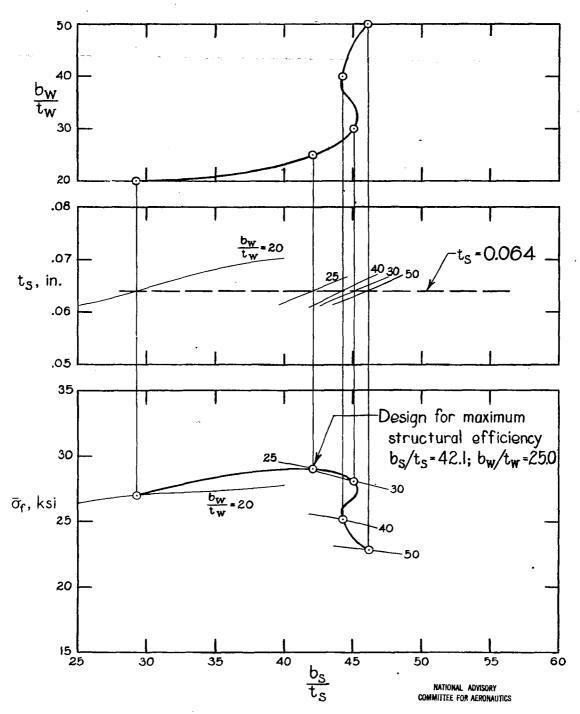


Figure II.-Plot for obtaining design for maximum structural efficiency.  $P_i = 3.0 \text{ kips/inch}$ ; L=20 inches; c=1;  $t_s = 0.064 \text{ inch}$ ;  $t_w/t_s = 0.79$ .

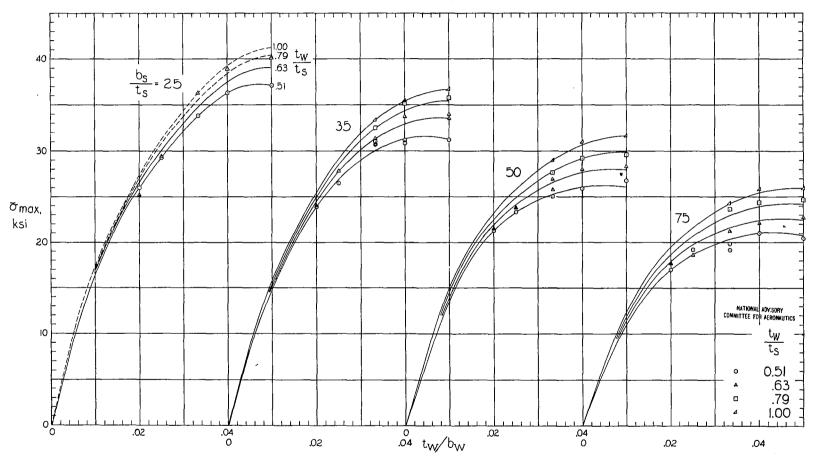


Figure 12.-Average stress at local failure for 24S-T aluminum-alloy flat compression panels with longitudinal Z-section stiffeners.  $b_{\rm F}/b_{\rm W}$ -0.4.

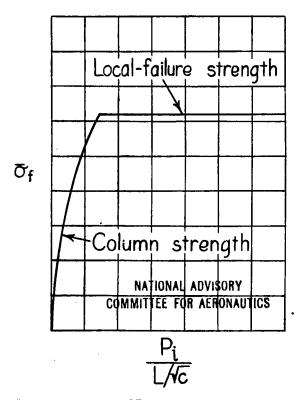


Figure 13. - Typical design curve for panels that do not buckle.

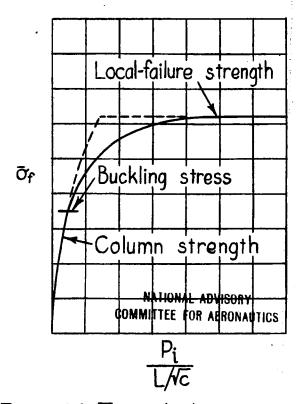


Figure 14.-Typical design curve for panels that buckle.

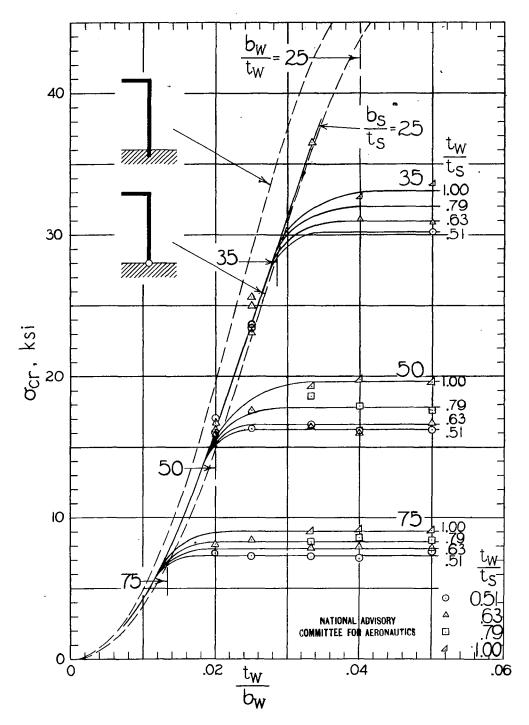


Figure 15.-Stress for local buckling of 24S-T aluminumalloy flat panels with Z-section stiffeners.  $b_F/b_W$ =0.4.

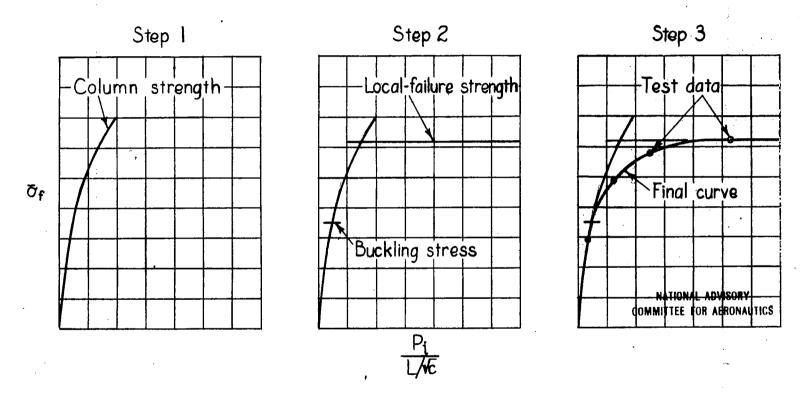


Figure 16.- Illustration of procedure used in preparation of design charts.

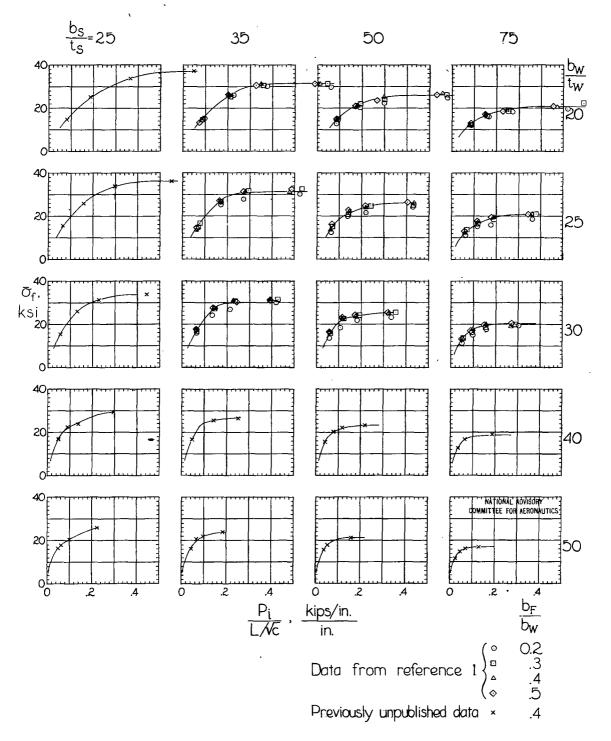


Figure 17. - Comparison of test data with design curves for 24S-T aluminum-alloy flat panels with Z-section stiffeners.  $t_W/t_S$ =0.51.

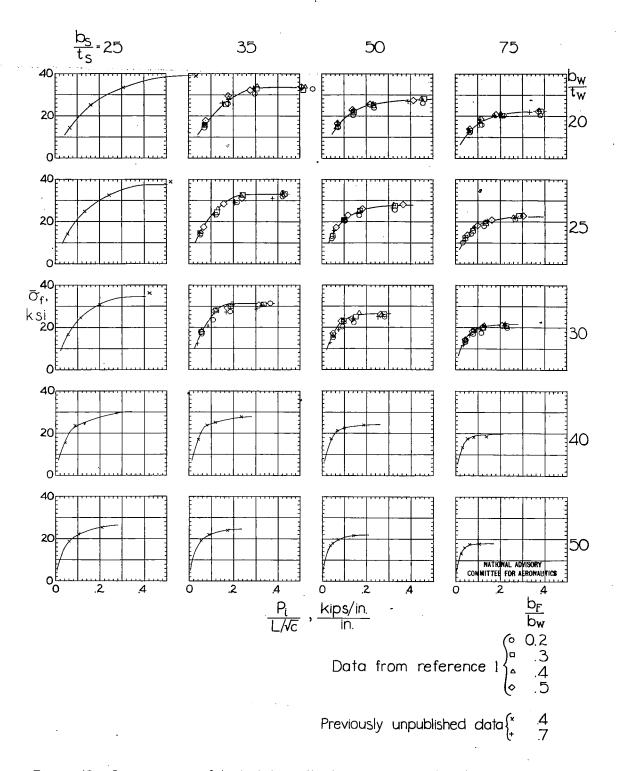


Figure 18.-Comparison of test data with design curves for 24S-T aluminumalloy flat panels with Z-section stiffeners.  $t_w/t_s$ = 0.63 .

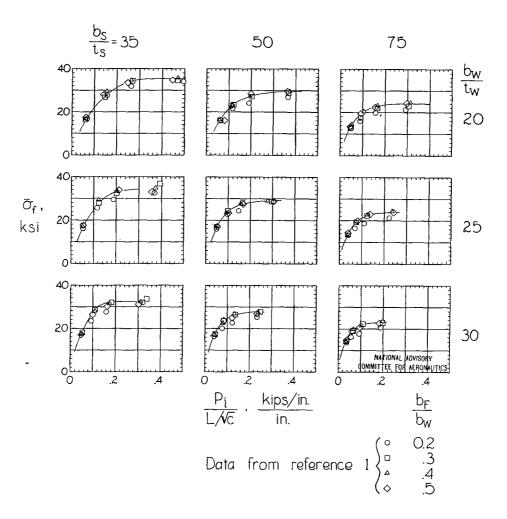


Figure 19.-Comparison of test data with design curves for 24S-T aluminum-alloy flat panels with Z-section stiffeners.  $t_W/t_S$ =0.79.

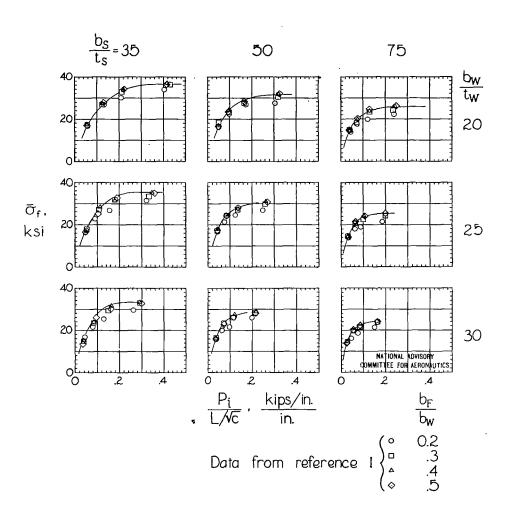


Figure 20.-Comparison of test data with design curves for 24S-T aluminum-alloy flat panels with Z-section stiffeners.  $t_W/t_s = 1.00$ .



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